

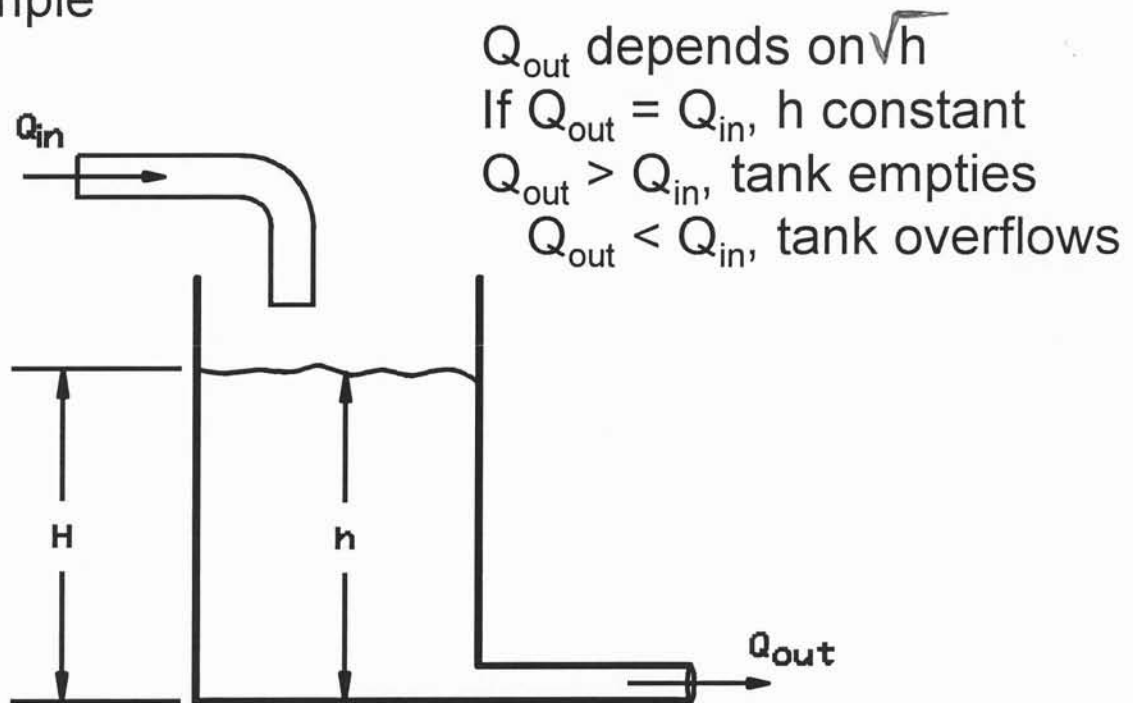
# Process Control Principles

## Fundamental control concepts

Regulate value of a quantity compared to a reference value. Reference called **setpoint**

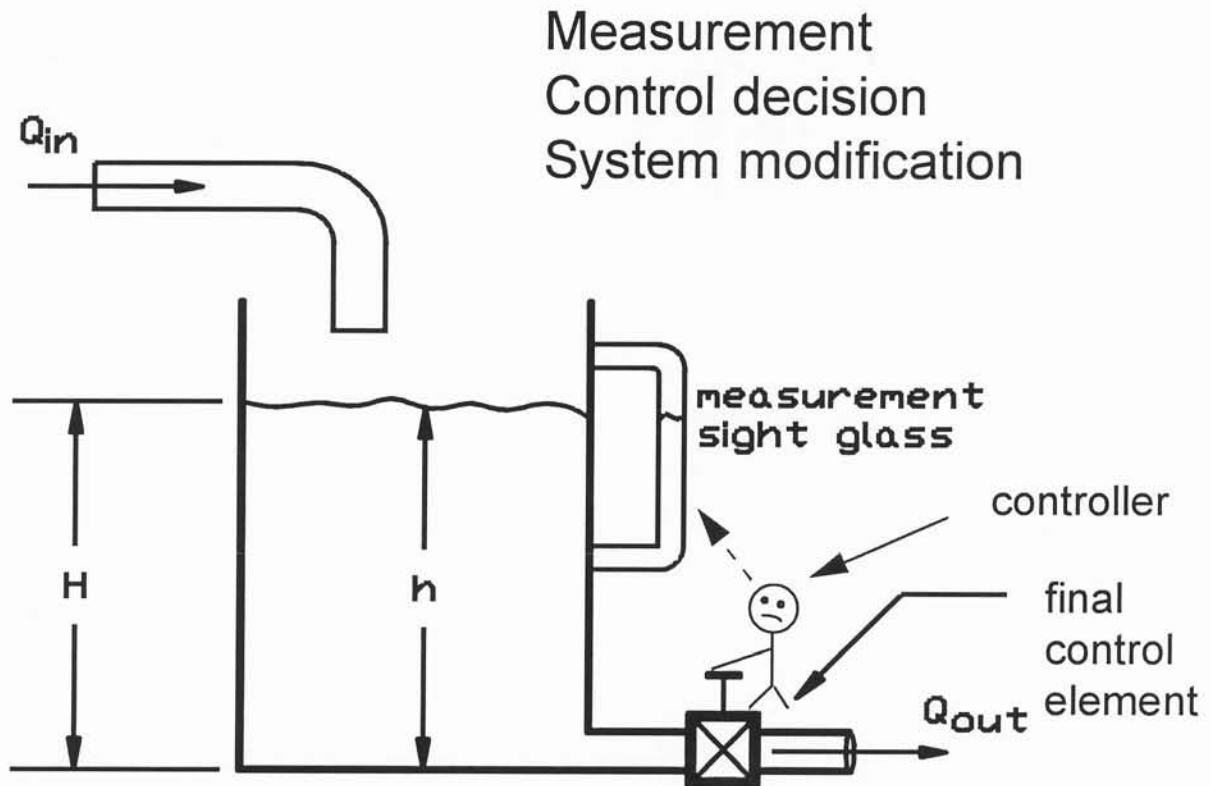
**Process** - a collection of equipment and materials that takes inputs and has outputs

Example



No reference value given

# Basic Control Elements

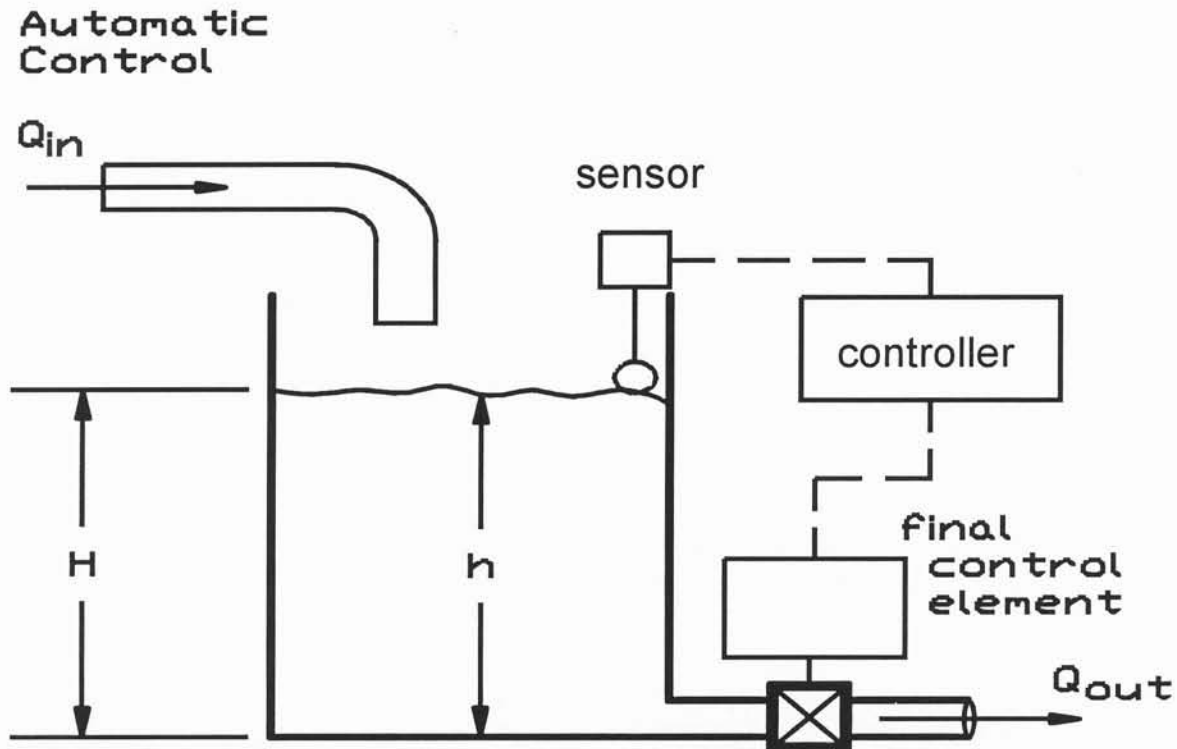


tank level,  $H$ , is reference (setpoint)  
 $h$  is the control variable.

Human is the controller, adjusts  
 $Q_{out}$  to maintain  $h$  as close as possible to  $H$

Example of regulator action

# Automatic Control Systems Automation



Use sensors and analog or digital electronics to monitor system.

## Elements of Automatic Control System

**Process** - single or multiple variable

**Measurement** - sensors / transducers

**Error detection** - compare  $H$  to  $h$

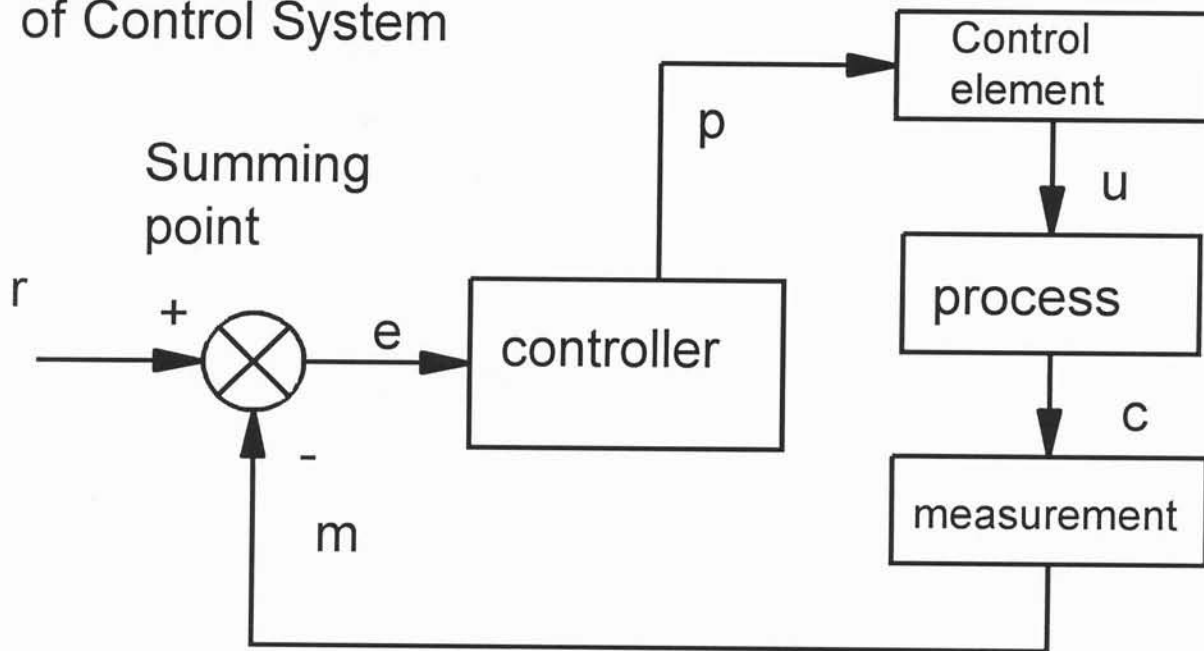
**Controller** - generate corrective action

**Final control element** - modify process

# Block Diagrams

Reduce complex systems to inputs and outputs

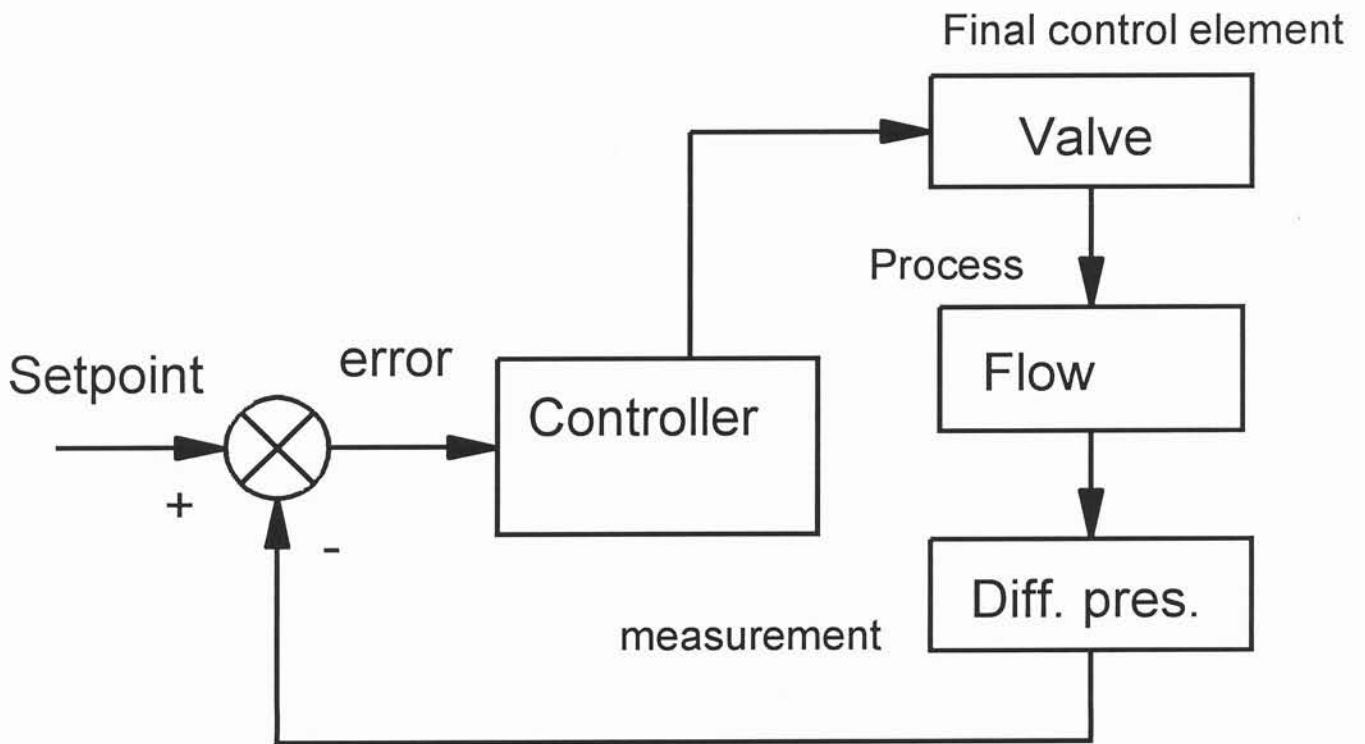
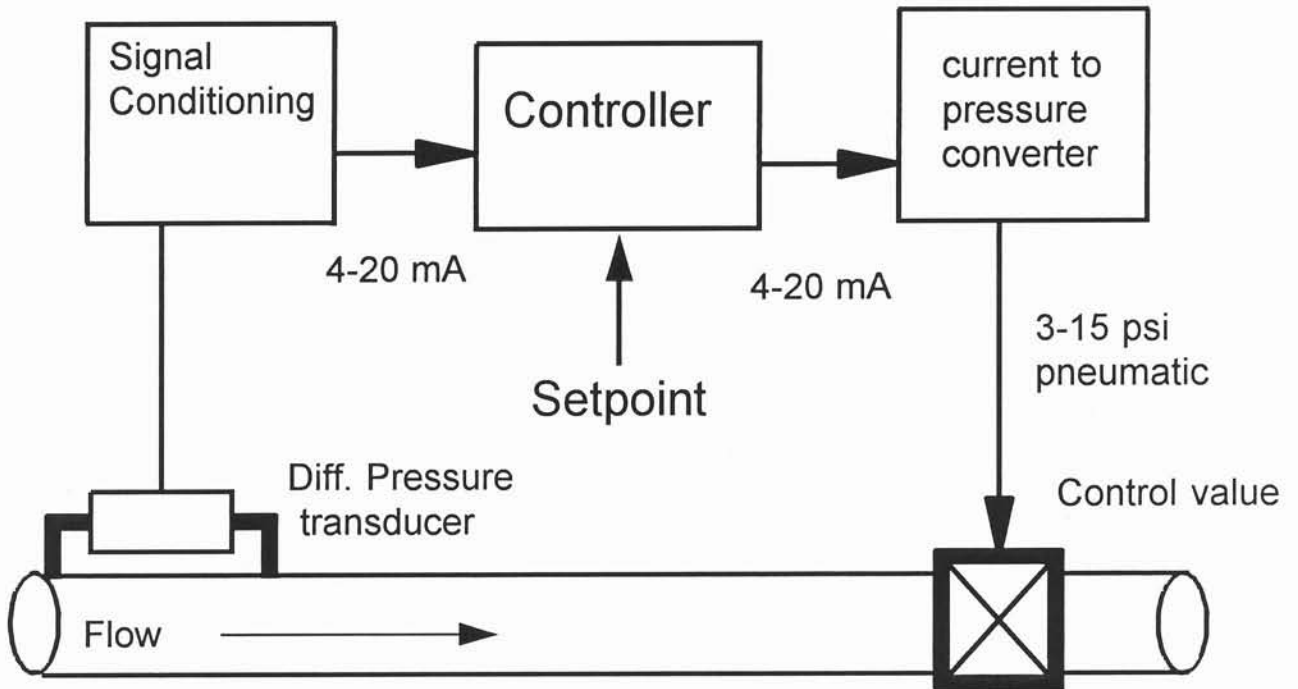
Block Diagram  
of Control System



Signals form a loop Control loop

$r$  = reference setpoint value  
 $c$  = control variable in the process  
 $m$  = measurement of control variable  
 $p$  = controller output  
 $u$  = control element change  
 $e = r - m$  (due to signs on summer)

# Practical System



# Control System Performance

System signal change over time so

$$e(t) = r - c(t)$$

Where  $e(t)$  = error as a function of time

$r$  = setpoint value

$c(t)$  = control variable as a function of time

## Control System Objectives

$e(t) = 0$  after changes or disturbances

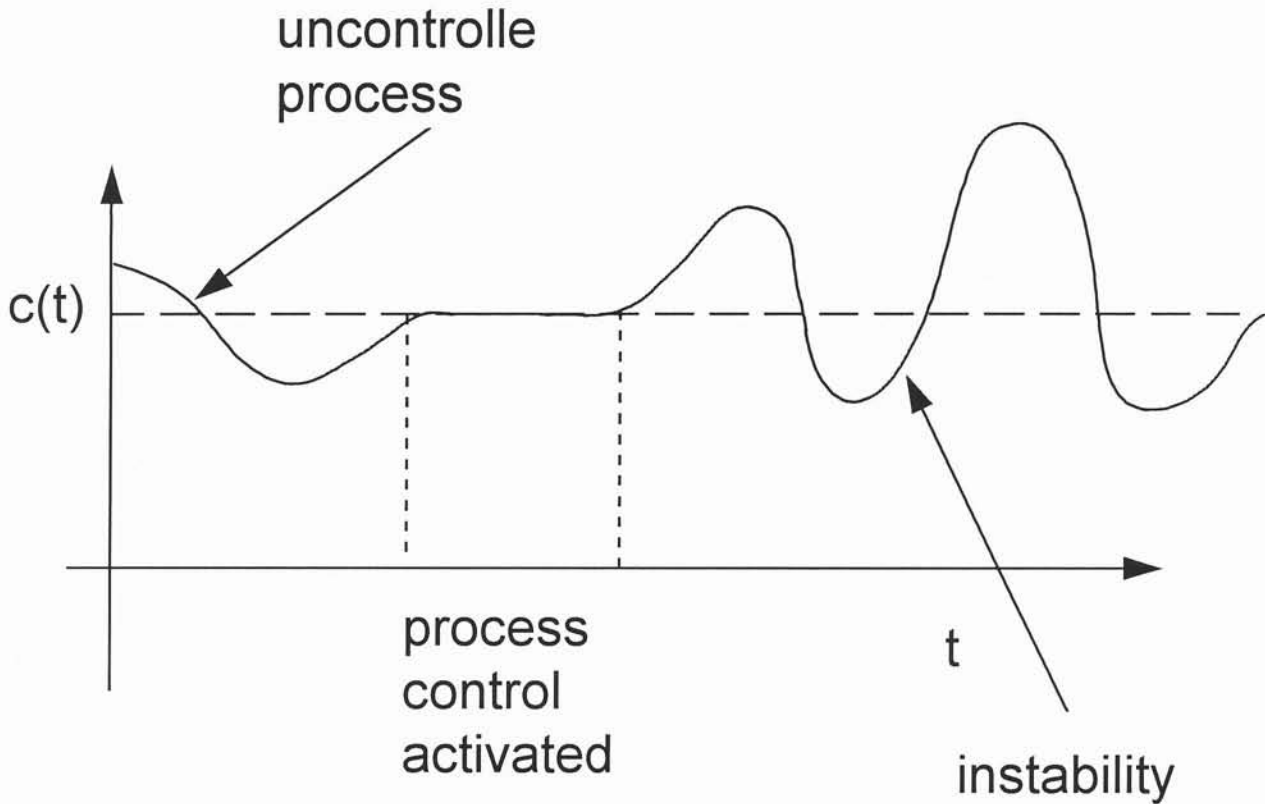
Stable  $c(t)$  after changes or disturbances

### Stability

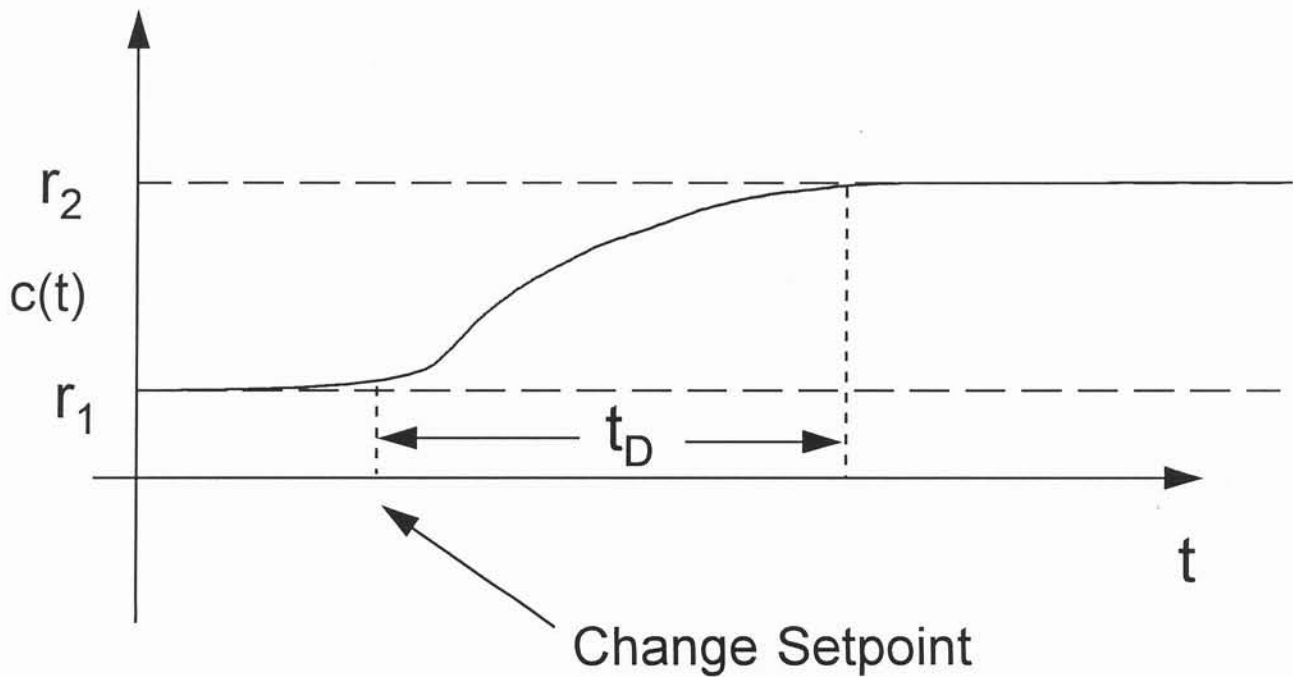
Steady-state regulation -  $e(t) = 0$  or within tolerances

Transient regulation - how does system perform under change

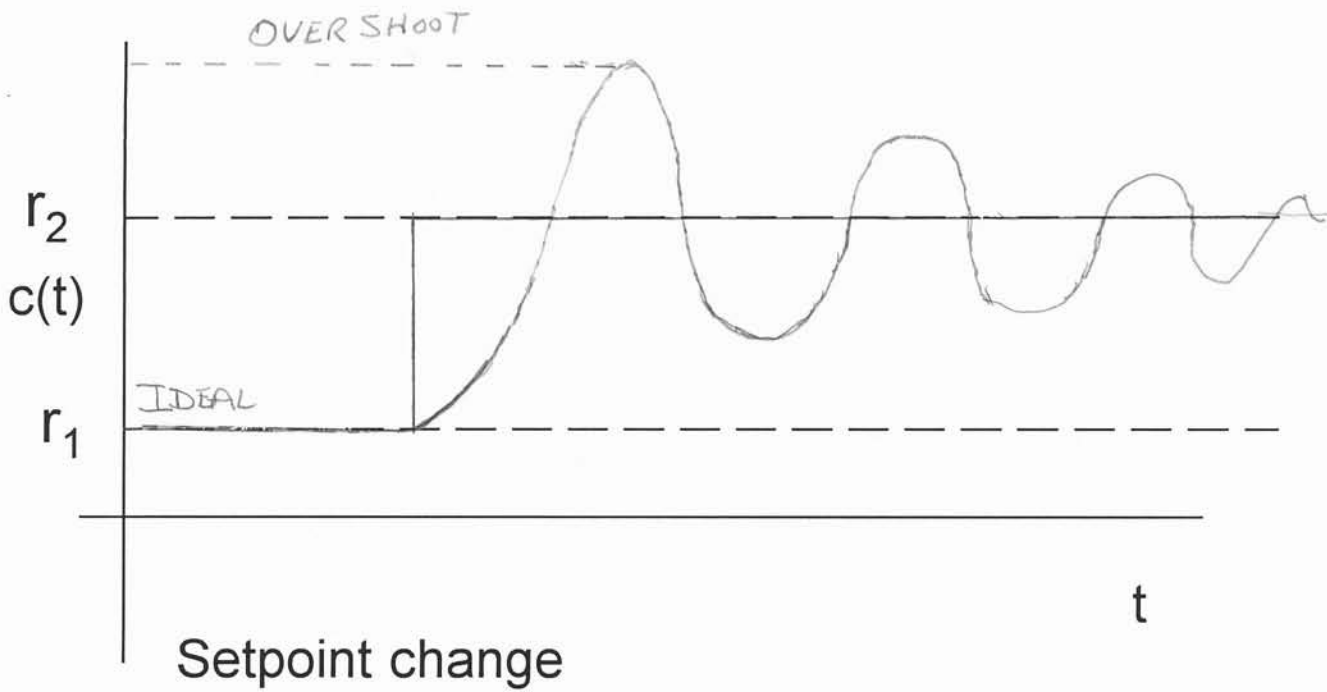
# Types of system response



## Damped Response



# Transient Responses





# Analog Measurement Error

Error determines accuracy

## Methods of determining accuracy

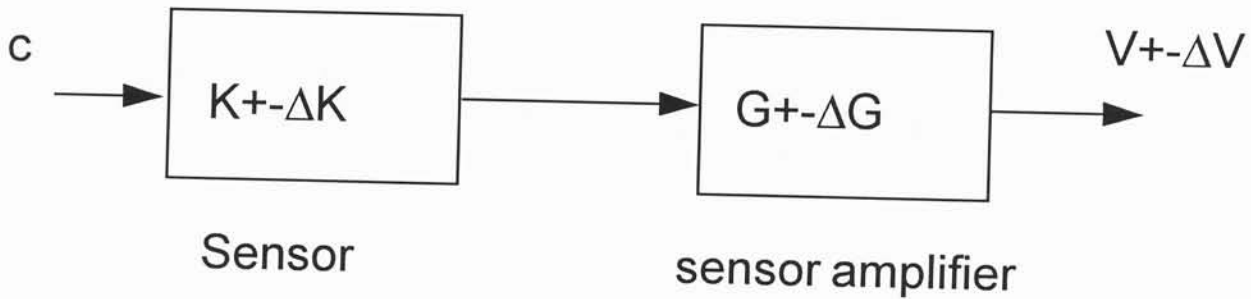
Measured Value: Reading  $\pm$  value  
100 psi  $\pm$  2 psi

Percent of Full Scale (FS): (FS Value) (%/100)  
Meter accurate to  $\pm$ 5% of FS)  
10 V scale  
error = 10 V ( $\pm$ 5%/100) =  $\pm$ 0.5V

Percentage of Span (Span = max-min)  
(Span)(%/100)  
P measurement  $\pm$ 3% of span  
20-50 psi  
error = (50-20)( $\pm$ 3%/100) =  $\pm$ 0.9 psi

Percentage of Reading  
reading of 2 V  $\pm$ 2%  
error = 2 V ( $\pm$ 2%/100) =  $\pm$  0.04 V

# System Accuracy Cummulative Error



$K$  = sensor gain     $G$  = amplifier gain  
 $V$  = sensor output voltage,  
 $\Delta G, \Delta V, \Delta K$  uncertainties in measurement

What is magnitude of  $\Delta V$ ?

With no error:  $V = (K)(G)c$

input

output

With error  $V \pm \Delta V = (K \pm \Delta K)(G \pm \Delta G)c$

Multiple out and simplify to get

$$\pm \frac{\Delta V}{V} = \pm \frac{\Delta G}{G} \pm \frac{\Delta K}{K} \quad \text{Where } \pm \frac{\Delta V}{V} = \text{normalized fractional uncertainty}$$

$$\pm \frac{\Delta G}{G} \pm \frac{\Delta K}{K} = \text{normalized fractional uncertainty}$$

# Combining the errors

## Use RMS (RSS)

$$\left(\frac{\Delta V}{V}\right)_{\text{Rms}} = \pm \sqrt{\left(\frac{\Delta G}{G}\right)^2 + \left(\frac{\Delta K}{K}\right)^2}$$

Notes:

$\frac{\Delta G}{G}$   $\frac{\Delta K}{K}$  are fractional error. Divide % to get.

$\frac{\Delta V}{V}$  is fractional RMS error Multiply by 100 to get percent.

Works on all formulas that are only multiplication and division.

Example: inverting OP AMP

$$v_o = v_{in} \frac{-R_f}{R_{in}}$$

$R_{in}$  &  $R_f$  5% Tolerance

$v_{in}$  Read with 2% accuracy

$$\frac{\Delta R_f}{R_f} = \frac{\Delta R_{in}}{R_{in}} = \frac{\pm 5\%}{100} = \pm 0.05$$

$$\frac{\Delta v_{in}}{v_{in}} = \frac{\pm 2\%}{100} = \pm 0.02$$

$$\begin{aligned} \left[\frac{\Delta v_o}{v_o}\right]_{\text{Rms}} &= \pm \sqrt{\left(\frac{\Delta R_f}{R_f}\right)^2 + \left(\frac{\Delta R_{in}}{R_{in}}\right)^2 + \left(\frac{\Delta v_{in}}{v_{in}}\right)^2} \\ &= \pm \sqrt{(0.05)^2 + (0.05)^2 + (0.02)^2} \\ &= \pm 0.073 \text{ OR } \pm \underline{\underline{7.3\%}} \text{ ANS} \end{aligned}$$

# Sensor Characteristics

**Sensitivity** - Change in output for change in input. Equals slope in linear device

**Hysteresis** - output different for increasing or decreasing input

**Resolution** - Smallest measurement a sensor can make.

**Linearity** - how close to a line is the I/O relationship

$$c_m = m(c) + c_0$$

Where  $c$  = measured control variable

$m$  = slope

$c_0$  = offset (y intercept)

$c_m$  = sensor output

## Example

Finding  $m$  and  $c_0$  from data points

$$y - y_1 = m(x - x_1) \quad m = \frac{y_2 - y_1}{x_2 - x_1}$$

Sensor has a linear resistance change of 100 to 195 ohms, as temperature changes from 20 - 120 C.

Find the sensor I/O relationship

Define Points  $(x_1, y_1) = (20^\circ\text{C}, 100\Omega)$   $x = \text{input}$   
 $(x_2, y_2) = (120^\circ\text{C}, 195\Omega)$   $y = \text{output}$

Slope  $m = \frac{195\Omega - 100\Omega}{120^\circ\text{C} - 20^\circ\text{C}} = \frac{95\Omega}{100^\circ\text{C}} = 0.95 \Omega/^\circ\text{C}$

$$y - 100\Omega = 0.95\Omega/^\circ\text{C}(x - 20^\circ\text{C})$$

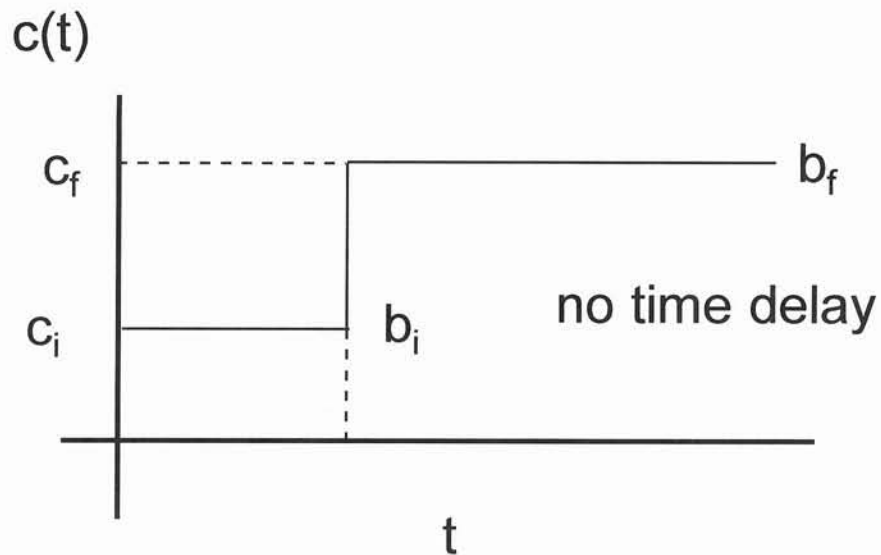
$$y = 0.95x - 0.95(20) + 100$$

$$y = \underline{0.95x + 81 \Omega/^\circ\text{C}} \quad \underline{\text{ANS}}$$

Plot to check points should be one line.

# Sensor Response

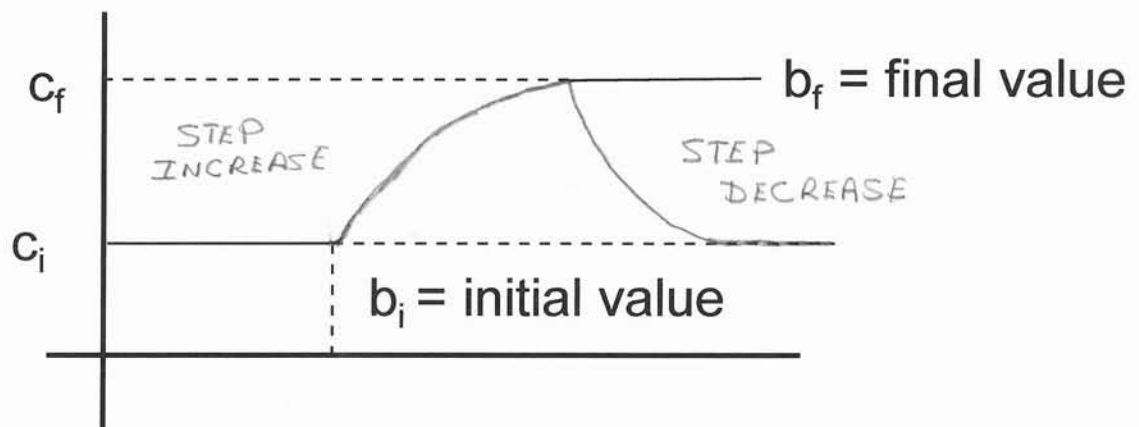
## First Order Response - ideal



## Step change in measured variable

## Practical sensor response

$c(t)$   $b(t)$  = sensor response function



# Modeling 1st Order Response

For step increase:

$$b(t) = b_i + (b_f - b_i) \cdot \left(1 - e^{-\frac{t}{\tau}}\right)$$

Where

$b_f$  = final sensor value

$b_i$  = initial sensor value

$t$  = time

$\tau$  = time constant of sensor

For step decrease

$$b(t) = (b_i - b_f) \cdot e^{-\frac{t}{\tau}}$$

Example: Step increase

$b_f = 4.0 \text{ V}$   $b_i = 2.0 \text{ V}$   $\tau = 0.0025 \text{ /s}$

Find time it takes for sensor to reach 90% of final value

$$b(t) = 2 + (4 - 2)(1 - e^{-t/0.0025})$$

$$b(t) = 2 + 2(1 - e^{-t/0.0025})$$

$b(t)$  at 90% of 4.0V = 3.6V

$$3.6 = 2 + 2(1 - e^{-t/0.0025})$$

$$\frac{3.6 - 2}{2} = 1 - e^{-t/0.0025}$$

$$0.8 - 1 = -e^{-t/0.0025}$$

$$+0.2 = +e^{-t/0.0025}$$

$$\ln(+0.2) = \ln(e^{-t/0.0025})$$

$$-1.609 = -t/0.0025$$

$$1.609(0.0025) = t$$

$$0.004 \text{ s} = t$$

4ms = t      ANSW

## Example: Step decrease

The sensor initial output is 1.0 V how long does it take to change to 0.2 V if the time constant of the sensor is 0.1/s.

$$b_i = 1.0 \text{ V} \quad b_f = 0.0 \text{ V} \quad \tau = 0.1 \text{ s} \quad b(t) = (b_i - b_f) e^{-t/\tau}$$
$$b(t) = 0.2 \quad 0.2 = 1 e^{-t/0.1}$$
$$\ln(0.2) = -t/0.1$$
$$-1.6094 = -t/0.1$$
$$1.6094(0.1) = t$$
$$\underline{\underline{0.1609 \text{ s}}} = \underline{\underline{t}} \quad \underline{\underline{\text{ANSW}}}$$

## Significant Digits in measurement and design

In measurement: readable output of instruments  
resolution of sensors and transducers.

Calculation using measurements: Truncate calculator answers to match significant digits of measurements and readings,



## Significant digit examples

Compute power based on the following measured values. Use correct number of significant digits.

$$I = 3.25 \text{ A} \quad V = 117.8 \text{ V} \quad P = V(I)$$

3.25 A     3 significant digits

117.8 V    4 significant digits

calculator 382.85 W

Truncate to 3 significant digits    **P = 383 W**

Significant digits not factor in design calculations  
Device values assumed to have no uncertainty.

Compute the current flow through a resistor that has a measured R of 1.234 k $\Omega$  and a voltage drop of 1.344 Vdc.

R = 1.234 k $\Omega$     4 significant digits

V = 1.344 V        4 significant digits

$$I = (1.344)/(1.234 \times 10^3) = 1.089 \text{ mA} \quad 4 \text{ digits}$$

# Basic Statistics

Arithmetic Mean ( Central tendency)

$$\bar{x} = \frac{\sum_{(i=1)}^n x_i}{n}$$

Where  $x_i$  = i th data measurement  
 $n$  = total number of measurements taken  
 $\bar{x}$  = mean value

Variance ( Measure of data spread from mean)

$$d_i = (x_i - \bar{x})^2 \quad \sigma^2 = \frac{\sum_{(i=1)}^n d_i}{n - 1}$$

$\sigma^2$  = variance of data

Standard Deviation

$\sigma$  = standard deviation

$$\sigma = \sqrt{\frac{\sum_{(i=1)}^n d_i}{n - 1}}$$

## Statistics Example

A 1000 ohm resistor is measured 10 times the following readings were taken

Test #	Reading	Test #	Reading
1	1016	6	1011
2	986	7	997
3	981	8	1044
4	990	9	991
5	1001	10	966

Find the mean, variance and standard deviation of the tests. What is the most likely value for the resistor to have?

$$\bar{X} = \frac{(1016 + 986 + 981 + 990 + 1001 + 1011 + 997 + 1044 + 991 + 966)}{10}$$

$$\bar{X} = 998.3 \Omega$$

$$d_1 = (1016 - 998.3)^2 \quad d_2 = (986 - 998.3)^2$$

$$d_1 = 313.29$$

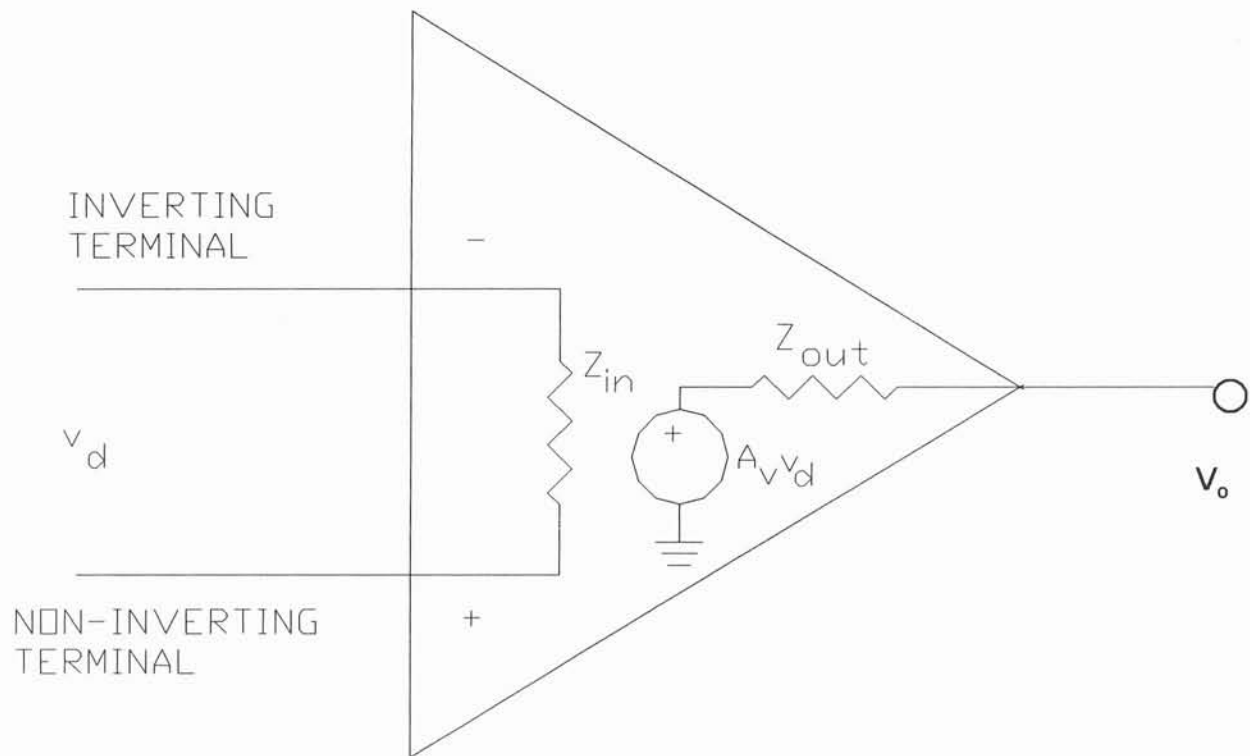
$$d_2 = 151.3$$

$$\sigma^2 = \frac{\sum_{i=1}^{10} d_i}{10-1} = 465.3 \quad \sigma = \sqrt{\sigma^2} = \sqrt{465.3} = 21.6$$

ALL VALUES SHOULD BE WITHIN  $\pm 3\sigma$

Most likely value for R is  $\bar{X}$  so 998.3  $\Omega$

# Ideal OP AMP Characteristics



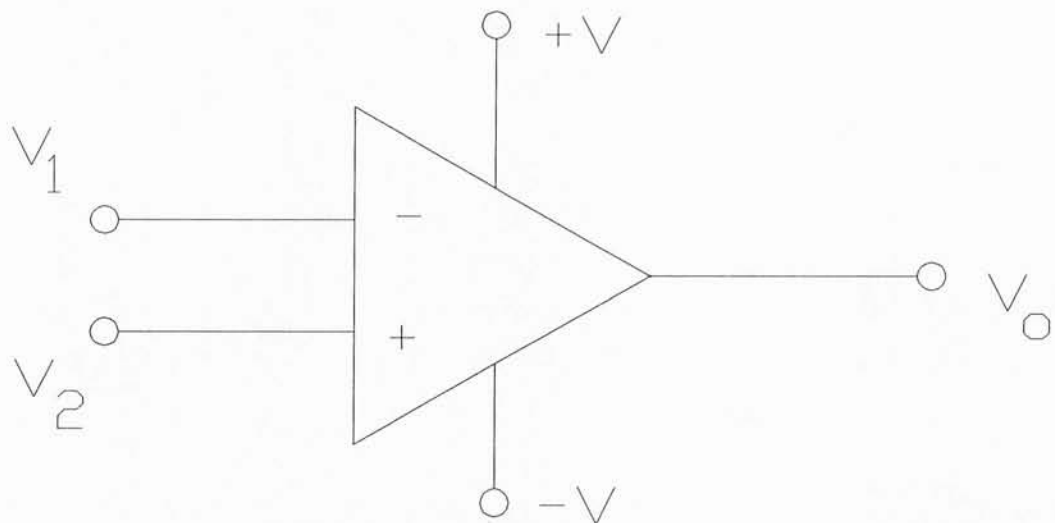
- 1.) infinite  $Z_{in}$
- 2.)  $Z_{out} = 0$  zero output impedance
- 3.) Bandwidth infinite Gain is constant for all frequencies
- 4.) With  $V_d = 0$ ,  $V_o = 0$  (No offset voltage)
- 5.) Infinite voltage gain ( $A_v$ )
- 6.) Instant recovery from saturation
- 7.)  $I_{in} = 0$  (due to infinite  $Z_{in}$ )

1.

## OPERATIONAL AMPLIFIERS (OP AMP)

VOLTAGE AMPLIFIER DESIGNED FOR USE IN  
ANALOG COMPUTERS.

Symbol for basic OP AMP



Two Inputs

$V_1$  Inverting input

$V_2$  Non-inverting input

$$V_1 > 0 \quad V_0 < 0$$

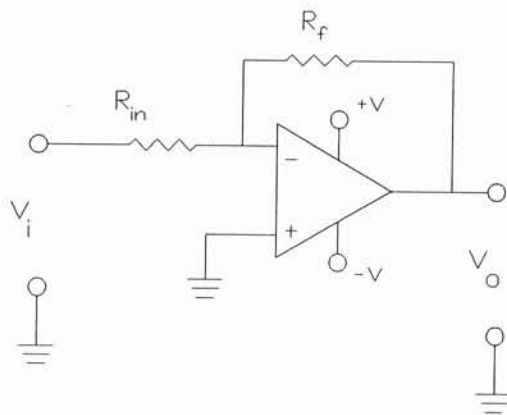
$$V_2 > 0 \quad V_0 > 0$$

Requires bipolar power supplies

OP AMPS ARE DIRECT COUPLED (dc) AMPLIFIERS AND CAN  
AMPLIFY dc & ac SIGNALS SIMULTANEOUSLY

# Basic Amplifier Circuits

## Inverting Voltage Amplifier



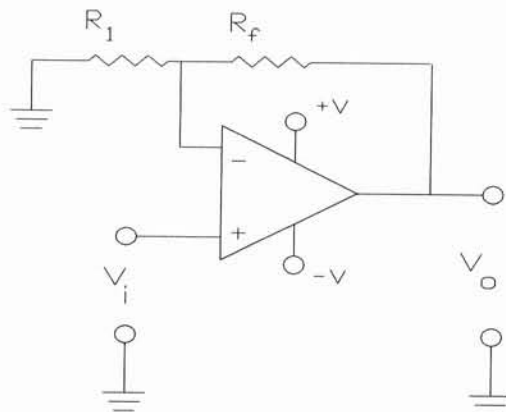
$$A_v = \frac{-R_f}{R_{in}}$$

$$V_o = V_i \left( \frac{-R_f}{R_{in}} \right)$$

$V_o$  Limited by  $\pm V$

Large  $A_v$  causes  $V_o = V$  (saturation)

## Non-Inverting Voltage Amplifier



$$A_v = \left( 1 + \frac{R_f}{R_{in}} \right)$$

$$V_o = V_i \left( 1 + \frac{R_f}{R_{in}} \right)$$

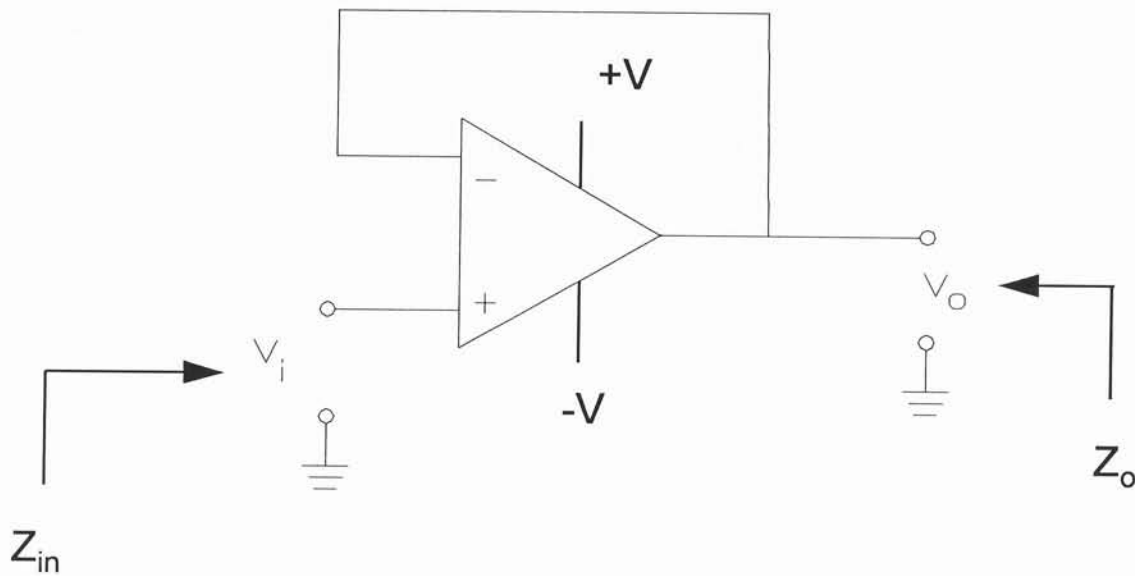
$R_{in} = R_{in}$  of OP AMP

$A_v$  has minimum value of 1

## Other useful OP AMP circuits

Impedance buffers (Voltage follower)

Used where high  $Z_{in}$  needed



### Characteristics

#### Practical Circuit (LM741)

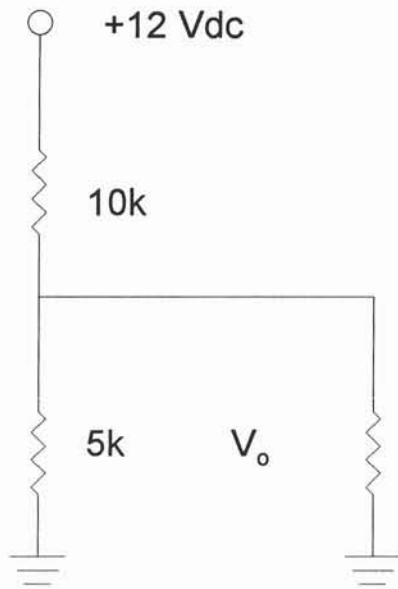
$$\begin{aligned}A_v &= 1 \\Z_{in} &= 1 \text{ MW} \\Z_o &= 10 \text{ W}\end{aligned}$$

#### Ideal

$$\begin{aligned}A_v &= 1 \\Z_{in} &= \text{infinite} \\Z_o &= 0\end{aligned}$$

Circuit causes minimum loading on previous stage

## Example: Buffered voltage divider circuit



Voltage divider formula only valid for infinite load resistance

Find  $V_o$  under load

NO LOAD  $V_o = \frac{5k\Omega}{10k\Omega + 5k\Omega} (12V)$

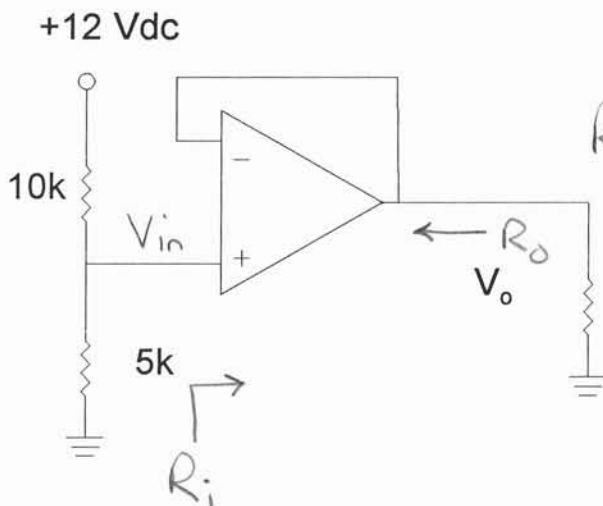
$V_o = \underline{4Vdc}$

WITH LOAD  $5k = R_L$

$R_L || 5k\Omega = 5k\Omega || 5k\Omega = 2.5k\Omega$

$V_o = \frac{2.5K}{10K + 2.5K} (12V) = \underline{\underline{2.4V}}$

Add impedance buffer



ASSUME LM741  $R_i = 1m\Omega$

$A_V = 1$  so  $V_{in} = V_o$   $R_o = 10\Omega$

$R_{eq} = R_i || 5k\Omega = 1m\Omega || 5k\Omega$

$R_{eq} = \frac{1m\Omega (5k\Omega)}{1m\Omega + 5k\Omega} = 4975\Omega$

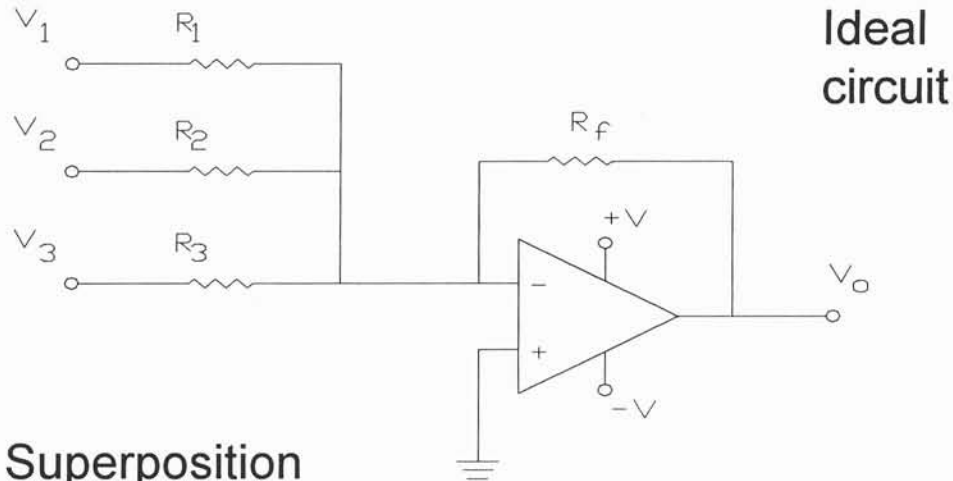
$V_{in} = \frac{4975\Omega}{10k\Omega + 4975\Omega} (12V) = \underline{\underline{3.987V}}$

$V_{in} = V_o = \underline{\underline{3.987V}}$



# Electronic Adding and Subtracting

## Inverting Summing Amplifier



Using Superposition

Gain  $v_1$   $\frac{-R_f}{R_1}$       Gain  $v_2$   $\frac{-R_f}{R_2}$       Gain  $v_3$   $\frac{-R_f}{R_3}$

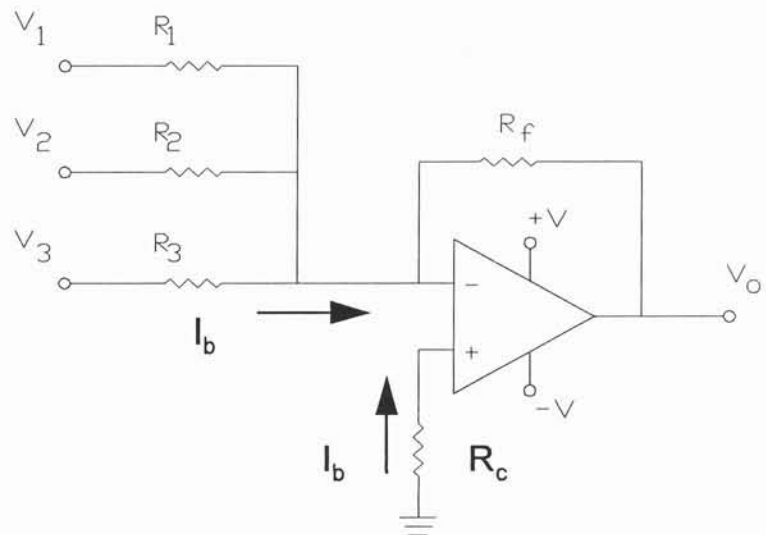
Total output 
$$v_o = -R_f \left( \frac{v_1}{R_1} + \frac{v_2}{R_2} + \frac{v_3}{R_3} \right)$$

Output is inverted sum of  $v_1$ ,  $v_2$ , and  $v_2$

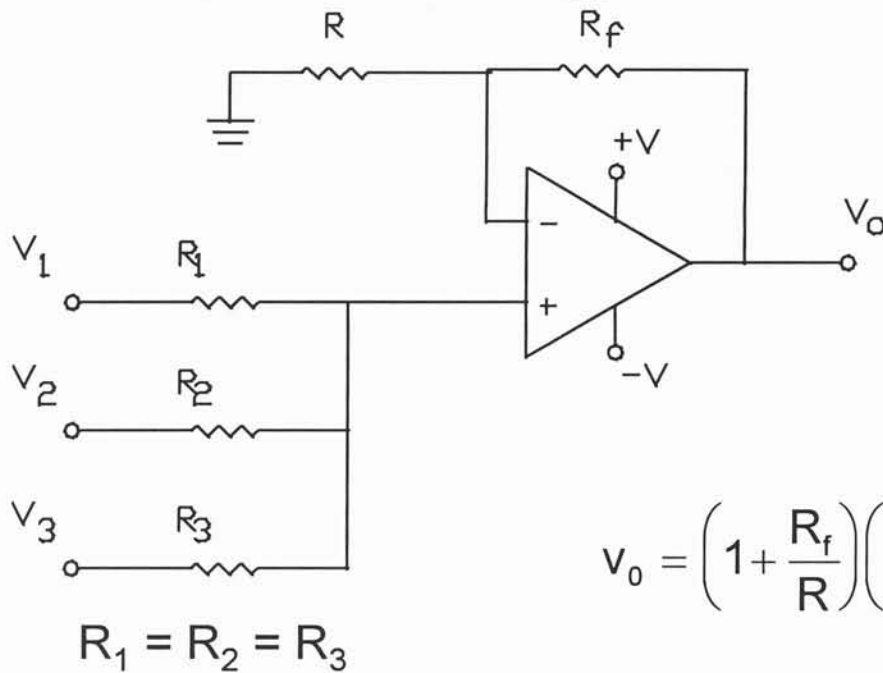
Improved circuit  
(non-ideal OP AMP)

bias compensation R

$$R_c = R_1 \parallel R_2 \parallel R_3 \parallel R_f$$



# Non-inverting Summing Amp



$$v_o = \left(1 + \frac{R_f}{R}\right) \left(\frac{v_1 + v_2 + v_3}{3}\right)$$

For any number, n, inputs.

$$v_o = \left(1 + \frac{R_f}{R}\right) \left(\frac{v_1 + v_2 + \dots + v_n}{n}\right)$$

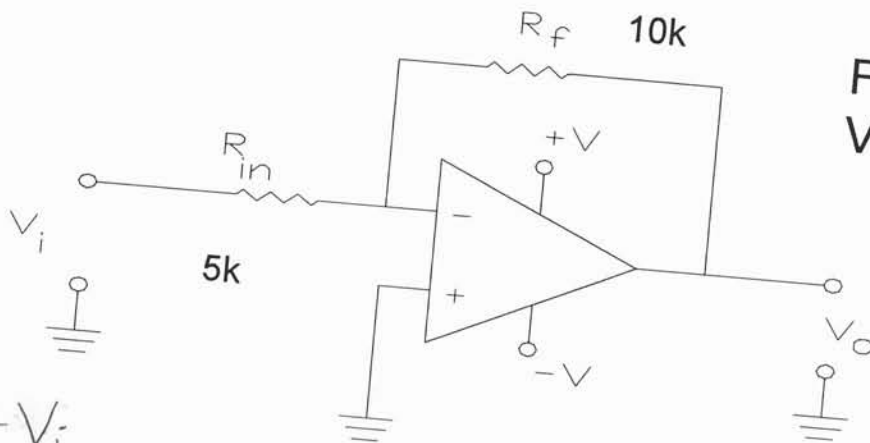
Assuming  $R_1 = R_2 = R_3 = \dots = R_n$

Example:  $n=3$ ,  $R_1 = R_2 = R_3 = 56k$   $R_f = 9k$   $R = 1k$   
 $V_1 = 0.5$  Vdc  $V_2 = 0.37$  Vdc  $V_3 = 0.8$  Vdc Find  $V_o$

$$v_o = \left(1 + \frac{R_f}{R}\right) \left(\frac{v_1 + v_2 + v_3}{3}\right)$$

$$v_o = \left(1 + \frac{9k}{1k}\right) \left(\frac{0.5 + 0.37 + 0.8}{3}\right) = 5.5667$$

# Inverting Amp Examples



Find  $V_o$  with  
 $V_{in} = 2 \text{ Vdc}$

$$V_o = \frac{-R_f}{R_{in}} V_i$$

$$V_o = \frac{-10 \text{ k}\Omega}{5 \text{ k}\Omega} V_i \rightarrow V_o = -2V_i \quad V_o = -2(2 \text{ V}) = \underline{\underline{-4 \text{ Vdc ANS}}}$$

Let  $R_f = 5 \text{ k}\Omega$ , find  $A_v$  &  $V_o$   $V_i = 2 \text{ Vdc}$

$$A_v = \frac{-R_f}{R_{in}} = \frac{-5 \text{ k}\Omega}{5 \text{ k}\Omega} = \underline{\underline{-1}} \text{ INVERTING VOLTAGE FOLLOWER}$$

$$V_{in} = 2 \text{ Vdc} \quad V_o = A_v V_i = -1(2) = \underline{\underline{-2 \text{ Vdc ANS}}}$$

Let  $R_f = 2.5 \text{ k}\Omega$  Find  $A_v$  &  $V_o$   $V_i = 2 \text{ Vdc}$

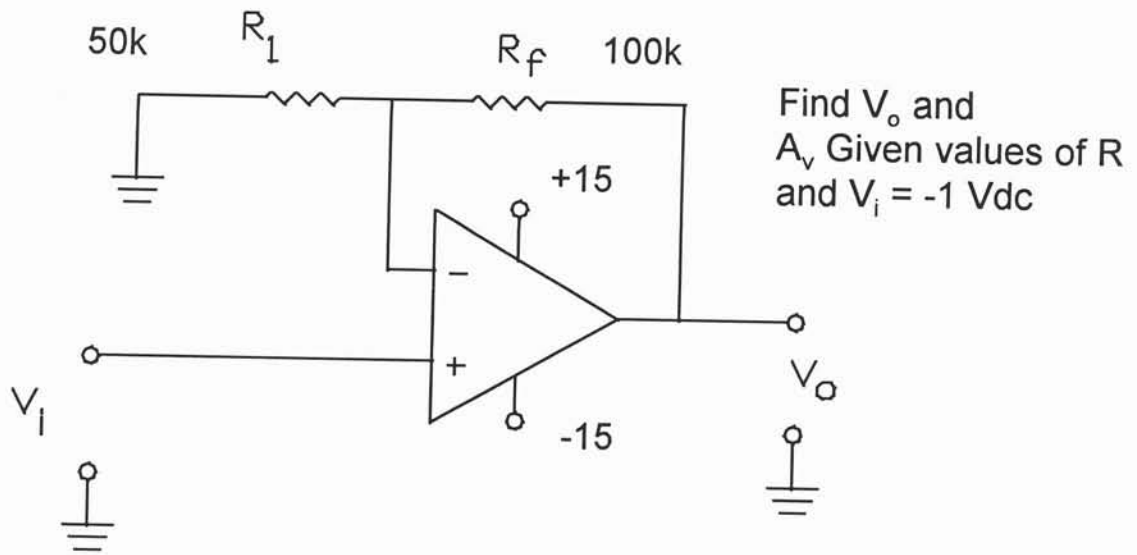
$$A_v = \frac{-R_f}{R_{in}} = \frac{-2.5 \text{ k}\Omega}{5 \text{ k}\Omega} = -0.5 \text{ Reduces INPUT VOLTAGE}$$

$$V_o = A_v V_i = -0.5(2 \text{ Vdc}) = \underline{\underline{-1 \text{ Vdc}}}$$

OUTPUT SMALLER THAN INPUT

Like Dividing  $V_i$  by 2

# Non-Inverting Amp Examples



$$V_o = V_i \left( 1 + \frac{R_f}{R_1} \right) = \left( 1 + \frac{100k\Omega}{50k\Omega} \right) V_i$$

$$V_o = -1(3) = \underline{\underline{-3 \text{ Vdc}}} \quad \underline{\underline{\text{ANS}}} \quad \underline{\underline{\text{NO SIGN CHANGE}}}$$

$$A_v = \left( 1 + \frac{R_f}{R_1} \right) = \left( 1 + \frac{100k}{50k} \right) = \underline{\underline{3}} \quad \underline{\underline{\text{ANS}}}$$

NOTE:  $R_{in}$  of OP AMP is  $\infty$  AMP WILL NOT REDUCE OUTPUT OF PREVIOUS STAGE

$V_i$  Rises to 6 Vdc, what is  $V_o$ ?

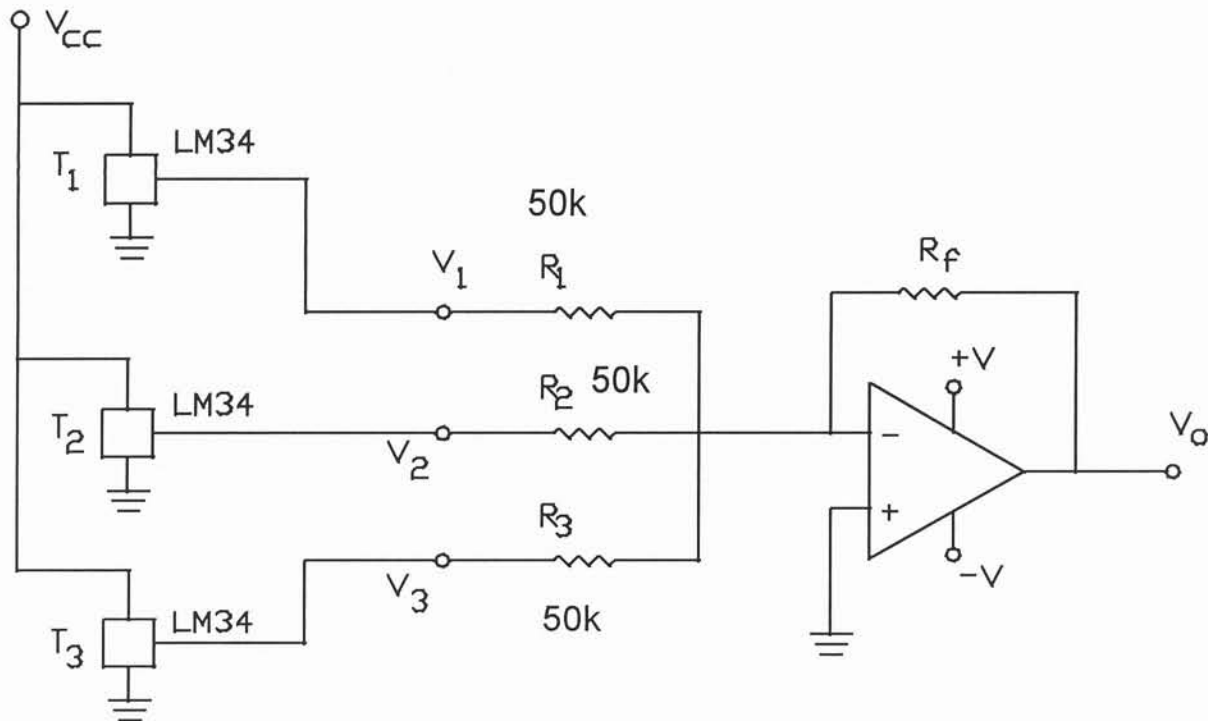
$$V_o = A_v V_i = 3(6) = 18 \text{ Vdc}$$

CAN'T HAPPEN! AMP SATURATES AT 13-15 Vdc  
Power Supply Limited

Ac signals distorted (Clipping)



# Averaging the Output of 3 Temperature Sensors



LM34 - temperature sensors. Gain = 10 mV/F  
 $T_1 = 50\text{ F}$   $T_2 = 45\text{ F}$   $T_3 = 50\text{ F}$

Average the temperature using a gain of -1 and -5. Find the value of  $R_f$  and  $V_o$  for each value of Gain

TO AVERAGE  $R_1 = R_2 = R_3 = 50\text{ k}\Omega$

$$V_o = -R_f \left( \frac{V_1}{R_1} + \frac{V_2}{R_2} + \frac{V_3}{R_3} \right)$$

SINCE  $R_1 = R_2 = R_3$

$$V_o = -R_f \left( \frac{V_1}{R_1} + \frac{V_2}{R_1} + \frac{V_3}{R_1} \right)$$

## Averaging Example continued

$$V_o = \frac{-3R_f}{R_1} (V_1 + V_2 + V_3) \text{ for 3 inputs}$$

for n inputs

$$V_o = \frac{-nR_f}{R_1} (V_1 + V_2 + V_3 + \dots + V_n)$$

for a gain of -1 find Relationship between  $R_1$  and  $R_f$

$$\frac{-nR_f}{R_1} = -1 \Rightarrow \boxed{R_f = \frac{R_1}{n}}$$

for any gain value  $A_v$   $\frac{-nR_f}{R_1} = A_v \Rightarrow \boxed{R_f = \frac{A_v R_1}{n}}$

find  $R_f$  and  $V_o$   $A_v = -1$

$$T_1 = 50^\circ\text{F} \quad V_1 = 10 \text{ mV}/^\circ\text{F} (T_1) = 10 \text{ mV}/^\circ\text{F} (50^\circ\text{F}) = \underline{0.5 \text{ V}}$$

$$T_2 = 45^\circ\text{F} \quad V_2 = 10 \text{ mV}/^\circ\text{F} (T_2) = 10 \text{ mV}/^\circ\text{F} (45^\circ\text{F}) = \underline{0.45 \text{ V}}$$

$$T_3 = 40^\circ\text{F} \quad V_3 = 10 \text{ mV}/^\circ\text{F} (T_3) = 10 \text{ mV}/^\circ\text{F} (40^\circ\text{F}) = \underline{0.40 \text{ V}}$$

$$R_1 = 50 \text{ k}\Omega \quad R_f = \frac{A_v R_1}{n} = \frac{-1(50 \text{ k}\Omega)}{3} = \underline{\underline{16.67 \text{ k}\Omega}} \text{ ANS}$$

$$V_o = \frac{-nR_f}{R_1} (V_1 + V_2 + V_3) = - \frac{(16.67 \text{ k}\Omega)}{50 \text{ k}\Omega} (0.5 + 0.45 + 0.4)$$

$$V_o = \underline{\underline{-0.45 \text{ V}}}$$

$$\text{AVERAGE} = \frac{0.5 + 0.45 + 0.4}{3} = 0.45$$

Checks

## Averaging Example continued

Part 2 Find  $R_f$  and  $V_o$  for  $A_v = -5$

Find Value of  $R_f$

$$R_f = \frac{A_v R_1}{n} \quad n=3 \quad R_1 = 50 \text{ k}\Omega \quad A_v = 5$$

$$R_f = \frac{5(50 \text{ k}\Omega)}{3} = \underline{\underline{83.33 \text{ k}\Omega}} \text{ ANS}$$

$$V_o = -\frac{R_f}{R_1} (V_1 + V_2 + V_3) = -\frac{83.33 \text{ k}\Omega}{50 \text{ k}\Omega} (0.5 + 0.45 + 0.4)$$

$$V_o = -1.667(1.35 \text{ V}) = \underline{\underline{-2.25 \text{ V}}} \text{ ANS}$$

$$\text{Check } 5 \left( \frac{0.5 + 0.45 + 0.4}{3} \right) = \underline{\underline{2.25}} \text{ checks}$$

NOTE:  $R_f$  is non-standard value

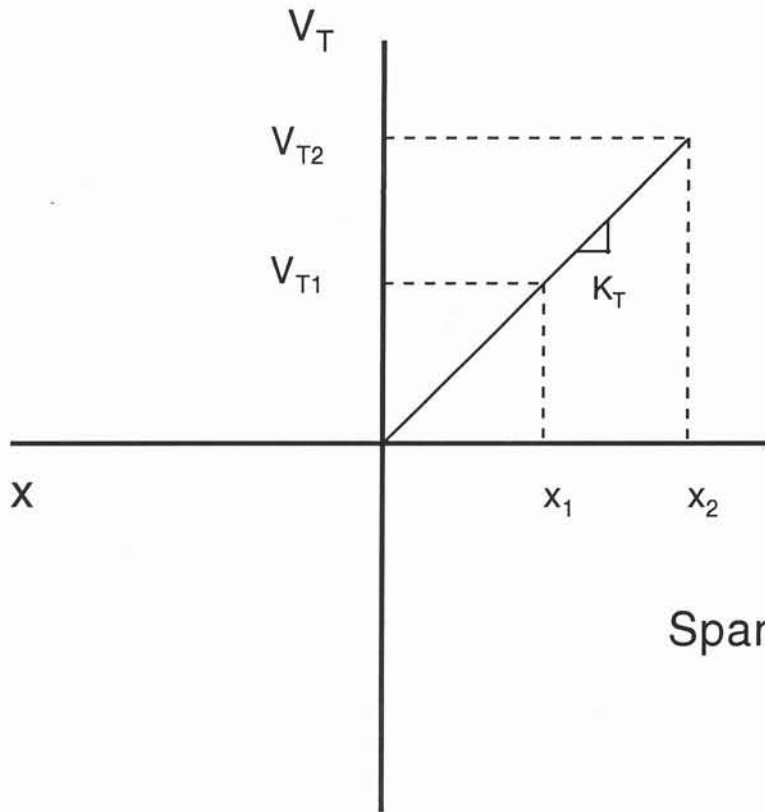
use closest standard value and series  
Potentiometer then calibrate.

Practical  $R_f$





# Scaling Linear Transducer Outputs

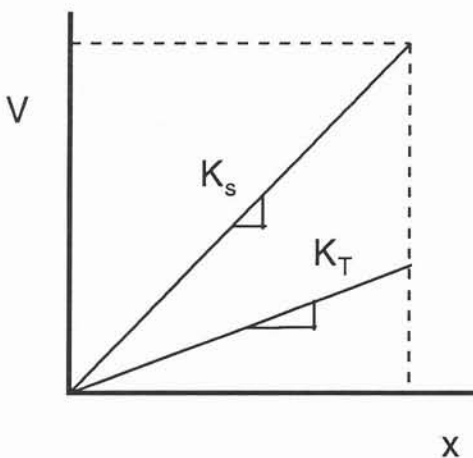


$x$  = transducer input  
 (measured value)  
 $V_T$  = transducer output  
 voltage  
 $K_T$  = transducer gain  
 (slope)  
 $K_s$  = scaler gain  
 $V_s$  = scaler output

Span = max. value - min. value

$$K_T = \frac{\text{output span}}{\text{input span}} = \frac{S_o}{S_i}$$

Case 1: no offset in transducer or scaler characteristics



Transducer gain formula:  $V_T = K_T x$

Required scaler gain:

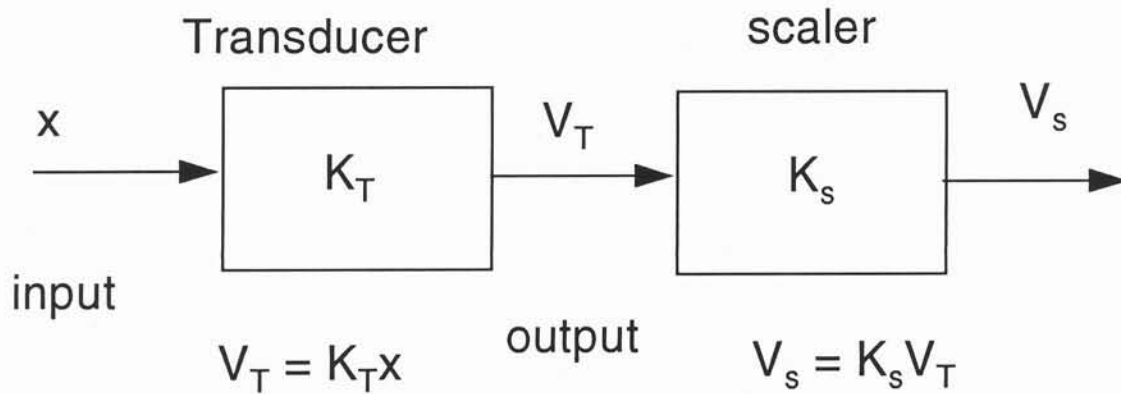
$$K_s = \frac{\text{desired span}}{\text{transducer span}} = \frac{S_d}{S_T}$$

scaler output formula

$$V_s = K_s \cdot K_T \cdot x$$

$$V_s = K_s \cdot V_T$$

## Block Diagram Case 1: multiply by constant



Example: Pressure transducer:  $P(x)$  range 0 - 50 psig  
 $V(V_T)$  range 0 - 1.25 Vdc

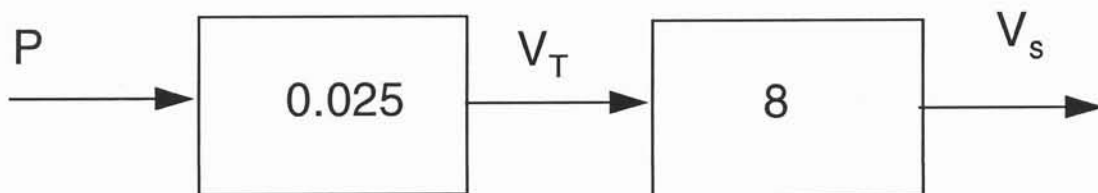
Desired output range: 0 - 10 Vdc

Find transducer gain, gain required to scale transducer to new range and draw the block diagram with calculated values shown.

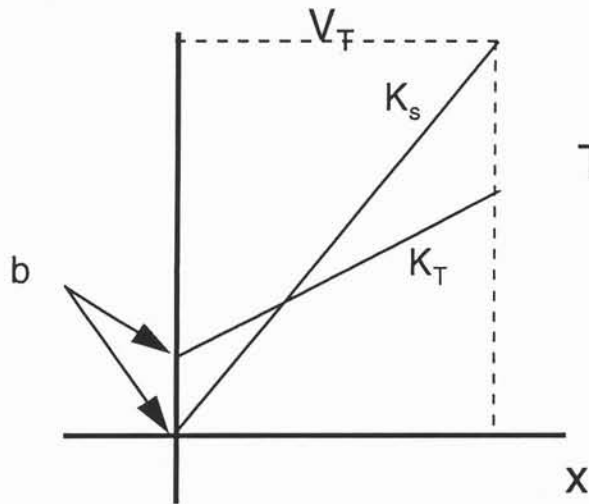
Find  $K_T$      $K_T = \frac{\text{output span}}{\text{input span}} = \frac{S_o}{S_i}$      $K_T = \frac{1.25 - 0 \text{ V}}{50 - 0 \text{ psig}} = 0.025 \text{ V / psig}$

Find scaler gain     $K_s = \frac{\text{desired span}}{\text{transducer span}} = \frac{S_d}{S_T}$

$$K_s = \frac{10 - 0 \text{ V}}{1.25 - 0 \text{ V}} = 8 \text{ V / V}$$



## Case 2: Transducer with offset - Output with no offset



$b =$  transducer offset

Transducer gain formula:

$$V_T = K_T x + b$$

Scaler gain formula

Must subtract offset

correct  
form of  
scaler output

$$V_s = K_s(K_T x + b) - K_s b$$

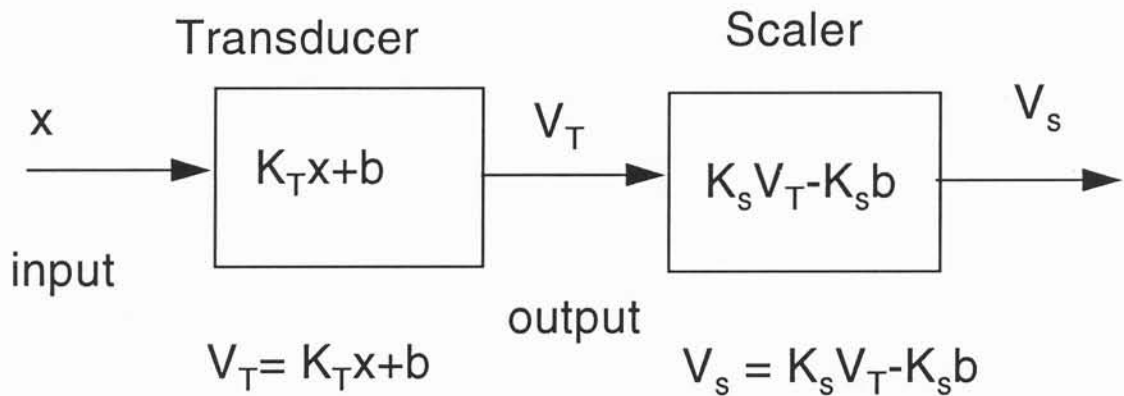
$$V_s = K_s K_T x + K_s b - K_s b$$

$$V_s = K_s K_T x$$

scaler equation

$$V_s = K_s V_T - K_s b$$

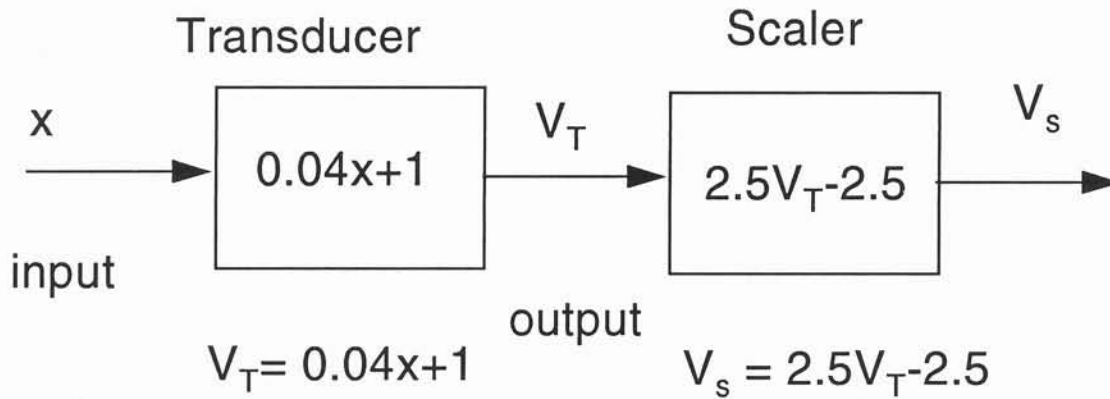
## Block diagram



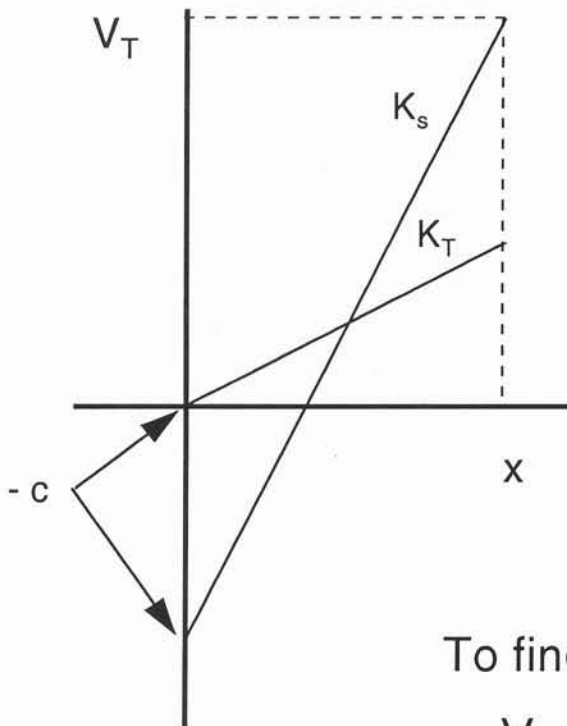


## Example: Case 2 continued

### Block Diagram



### Case 3: Transducer with no offset, Output offset



$c$  = scaler offset can be +- value

Transducer gain formula:

$$V_T = K_T x$$

Scaler formula must add a constant

$$V_s = K_s V_T + c$$

$$K_s = \frac{\text{desired span}}{\text{transducer span}} = \frac{S_d}{S_T}$$

To find  $c$ , use point slope form

$$V_s - V_{s1} = K_s (V_T - V_{T1})$$

Case 3: Example: Pressure transducer  
 range 0 - 25 psig (x)  
 0 - 1 V ( $V_T$ )

Desired range (-5 V to 5 Vdc) Find transducer gain and scaler gain formula. Draw the block diagram of the system

Find  $K_T$  
$$K_T = \frac{\text{output span}}{\text{input span}} = \frac{S_o}{S_i} \quad K_T = \frac{1 - 0 \text{ V}}{25 - 0 \text{ psig}} = 0.04 \text{ V / psig}$$

Find  $K_s$  
$$K_s = \frac{\text{desired span}}{\text{transducer span}} = \frac{S_d}{S_T}$$

$$\frac{S_d}{S_T} = \frac{5 - (-5) \text{ V}}{1 - 0 \text{ V}} = \frac{10}{1} = 10 \text{ V / V} = K_s$$

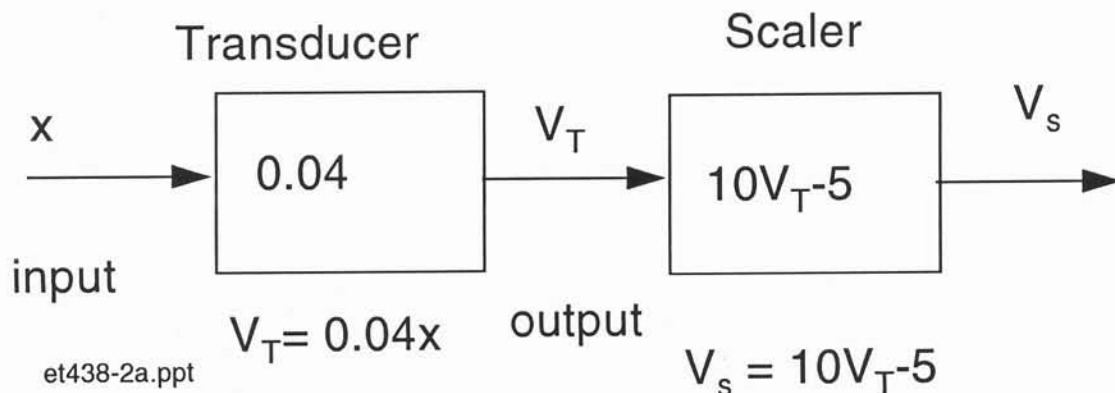
Use point-slope form of line to find the value of c

$V_s - V_{s1} = K_s(V_T - V_{T1})$  ( $V_{T1}, V_{s1}$ ) = (1, 5) can use either point in pair defining range

$$V_s - 5 = 10(V_T - 1)$$

$V_s = 10V_T - 5$  scaler gain formula  
 $c = -5$  in this case

Block Diagram



# Example 1

Range of linear temperature transducer is 32- 212 F with a transducer gain of 10 mV/F. The desired output of the transducer for the range of temperature is 0 - 10 Vdc. Find the gain formula.

$K_T = 0.01 \text{ V/F}$  FIND SPAN AT  $T_1 = 32\text{F}$   $V_{T1} = 0.01 \text{ V/F} (32\text{F}) = 0.32 \text{ V}$

$S_T = V_{T2} - V_{T1} = 2.12 - 0.32 \text{ V}$

AT  $T_2 = 212\text{F}$   $V_{T2} = 0.01 \text{ V/F} (212\text{F}) = 2.12 \text{ V}$

$S_T = 1.8 \text{ V}$

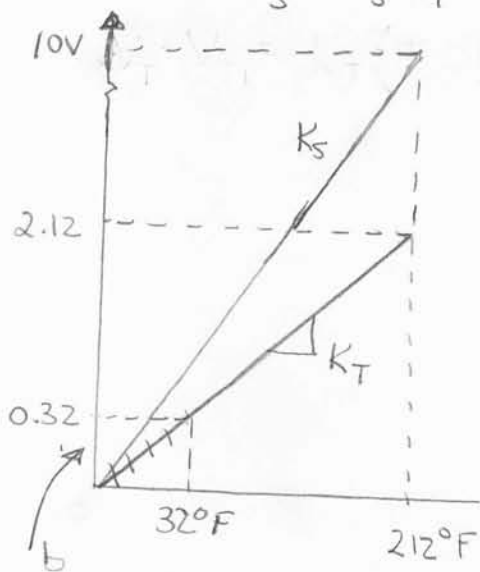
OUTPUT AT 32°F scaled to 0V

$S_d = \text{desired span}$   $S_d = 10 - 0 \text{ V} = 10 \text{ V}$   $V_T = K_T T + b$

$K_s = \frac{S_d}{S_T} = \frac{10 \text{ V}}{1.8 \text{ V}} = 5.56$

CASE 2 SCALING MUST REMOVE OFFSET

$V_s = K_s V_T - K_s b$  FIND  $b$  from Graph  $b = 0.32 \text{ V}$



$V_s = 5.56 V_T - 5.56(0.32) \text{ V/V}$

$V_s = 5.56 V_T - 1.779 \text{ V/V}$  check

at 32°F  $V_T = 0.32 \text{ V}$

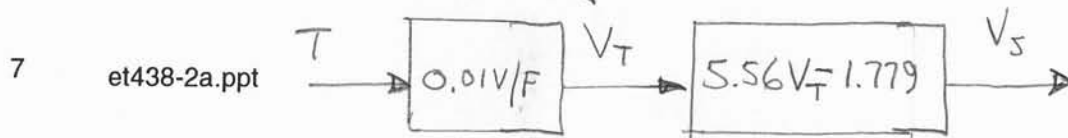
$V_s = 5.56(0.32) - 1.779 = 0 \text{ V}$

at 212°F  $V_T = 2.12 \text{ V}$

$V_s = 5.56(2.12) - 1.779 = 10.01 \text{ V}$

Checks

Block Diagram



## Example 2

Linear temperature transducer

Range:  $-20\text{ C}$  to  $50\text{ C}$   $K_T = 20\text{ mV/C}$

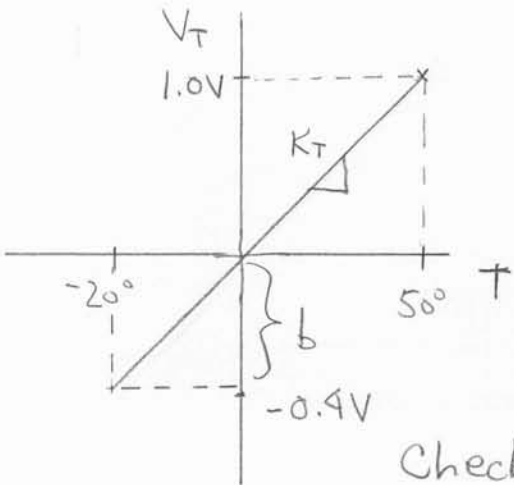
Desired range:  $0 - 5\text{ Vdc}$ .

Transducer output voltage is bipolar (+/-)

$$V_T \text{ at } -20^\circ\text{C} \quad V_{T1} = 20\text{ mV/C}(-20^\circ\text{C}) = -0.4\text{V}$$

$$V_T \text{ at } 50^\circ\text{C} \quad V_{T2} = 20\text{ mV/C}(50^\circ\text{C}) = 1.0\text{V}$$

SCALER FORMULA  $V_S = K_S V_T - K_S b$  Find  $b$  Graphically



$$K_S = \frac{S_d}{S_T} = \frac{(5-0)\text{V}}{1-(-0.4)} = 3.571\text{ V/V}$$

$$b = -0.4\text{V from plot}$$

$$V_S = 3.571 V_T - 3.571(-0.4)$$

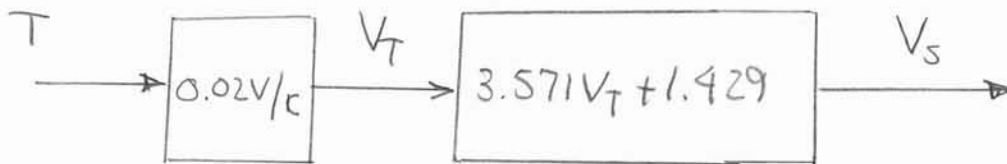
$$V_S = 3.571 V_T + 1.429\text{ V/V}$$

Check  $V_T = -0.4\text{V}$  at  $-20^\circ\text{C}$

$$V_0 = 3.571(-0.4\text{V}) + 1.429 \approx 0\text{V}$$

$V_T = 1\text{V}$  at  $50^\circ\text{C}$

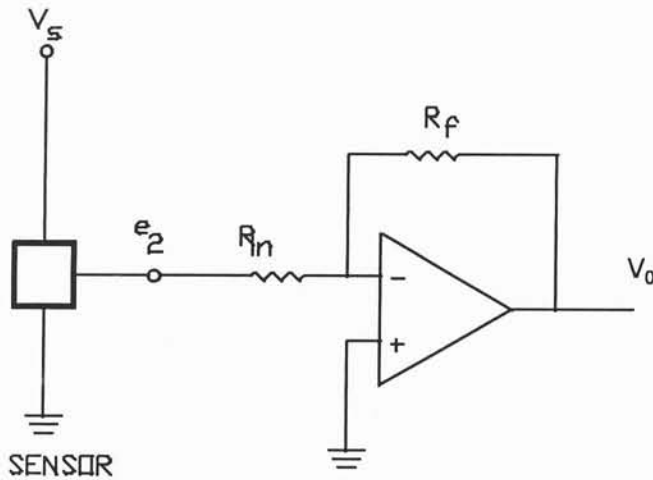
$$V_0 = 3.571(1) + 1.429 = 5.0\text{V}$$





# Practical Realization of gain formula

Use OP AMP circuits

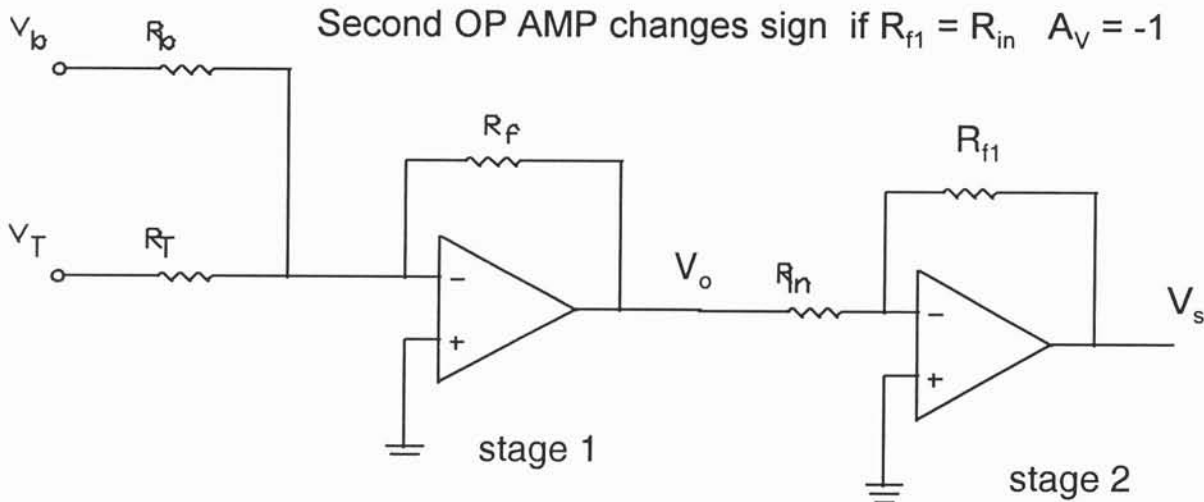


scaling without offset:  
use inverting or non-inverting  
amps to implement  $K_s$

For inverting

$$K_s = \frac{-R_f}{R_{in}}$$

For transducers with offset use inverting summation amps



Second OP AMP changes sign if  $R_{f1} = R_{in}$   $A_v = -1$

stage 1

overall gain

$$V_o = \frac{-R_f}{R_T} V_T + \frac{-R_f}{R_b} V_b \quad V_s = \left( \frac{-R_f}{R_T} V_T + \frac{-R_f}{R_b} V_b \right) (-1) = \frac{R_f}{R_T} V_T + \frac{R_f}{R_b} V_b$$

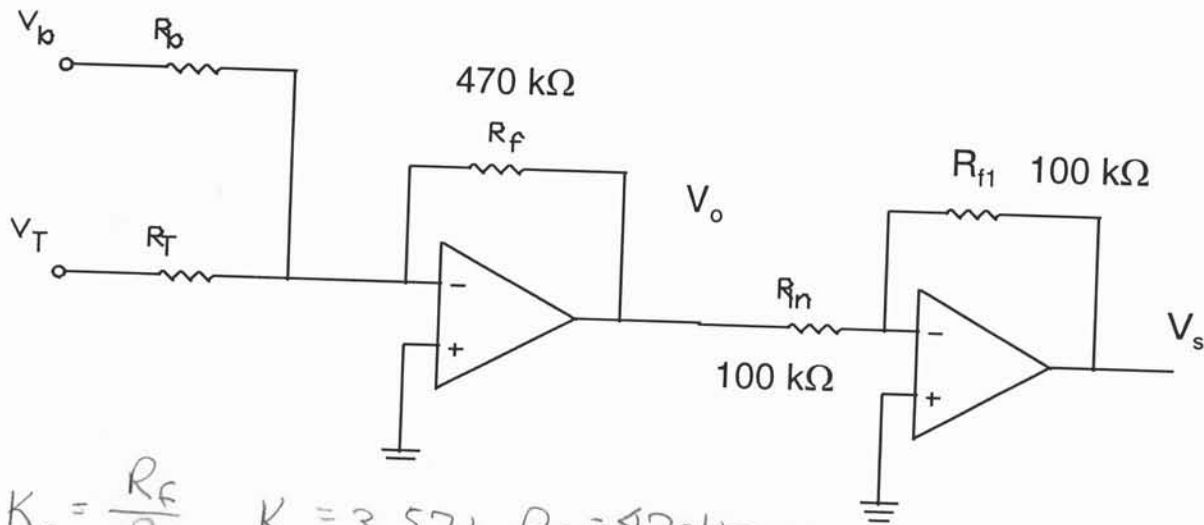
Gain formula with offset

$$V_s = (K_s V_T - K_s b) = \frac{R_f}{R_T} V_T + \frac{R_f}{R_b} V_b \quad \text{so} \quad K_s = \frac{R_f}{R_T} \quad K_s b = \frac{R_f}{R_b} V_b$$

Example: Design an OP AMP circuit that will implement the scaler equation from Example 2

Assume  $R_{f1} = R_{in} = 100 \text{ k}\Omega$   $R_f = 470 \text{ k}\Omega$

$V_s = 3.571V_T + 1.429$  scaler relationship for Example 2



$$K_s = \frac{R_f}{R_T} \quad K_s = 3.571 \quad R_f = 470 \text{ k}\Omega$$

SOLVE FOR  $R_T$   $R_T = \frac{R_f}{K_s} \quad R_T = \frac{470 \text{ k}\Omega}{3.571} = \underline{\underline{131.61 \text{ k}\Omega}}$

for  $R_T$  use standard value of potentiometer

$\frac{127 \text{ k}\Omega}{10 \text{ k}\Omega}$  calibrate with  $V_T = -0.4$  and  $1.0 \text{ V}$

$$K_{sb} = 1.429 \quad R_f > 0 \text{ AND } R_b > 0$$

$$K_{sb} = \frac{R_f}{R_b} V_b \quad \text{Assume } V_b = 2.0 \text{ V must be } > 0$$

$$1.429 = \frac{R_f}{R_b} (2) \Rightarrow \frac{1.429}{2} = \frac{R_f}{R_b} \quad R_f = 470 \text{ k}\Omega \text{ so}$$

$$0.7145 R_b = 470 \text{ k}\Omega \rightarrow R_b = \frac{470 \text{ k}\Omega}{0.7145} = \underline{\underline{657.80 \text{ k}\Omega}}$$

Simple case make  $\frac{R_f}{R_b} = -1$  so  $V_b = K_{sb}$

## Models of Components

### Types of components in controls systems

electrical  
mechanical  
liquid flow  
gas flow  
thermal

Behavior of systems defined by characteristics

Example: electrical

resistance	voltage
capacitance	current
inductance	charge
delay	

Resistance - amount of potential required to produce a unit current.

Capacitance - amount of charge required to make a unit change in potential.

Inductance - amount of potential required to make a unit change in rate of flow (current).

Delay (Dead-time) - time interval between signal appearing on input and response appearing on output.

## Defining Equations

Resistance

Static resistance (linear)

$$R = \frac{e}{i}$$

Dynamic Resistance (non-linear) Depends on the values of  $e$  and  $i$ .

$$R = \frac{\Delta e}{\Delta i} = \frac{de}{di}$$

Can estimate dynamic  $R$  with slope of tangent line at operating point.

$$R = \frac{\Delta e}{\Delta i} = \frac{e_2 - e_1}{i_2 - i_1}$$

## Capacitance

$$C = \frac{\Delta q}{\Delta e}$$

$I$  is rate of change of flow  
Columb/sec = Amp

$$C \cdot \Delta e = \Delta q$$

$$C \cdot \frac{\Delta e}{\Delta t} = \frac{\Delta q}{\Delta t} = i$$

$$i = C \cdot \frac{de}{dt}$$

Definition of capacitance

## Inductance

$$e = L \cdot \frac{\Delta i}{\Delta t} = L \cdot \frac{di}{dt}$$

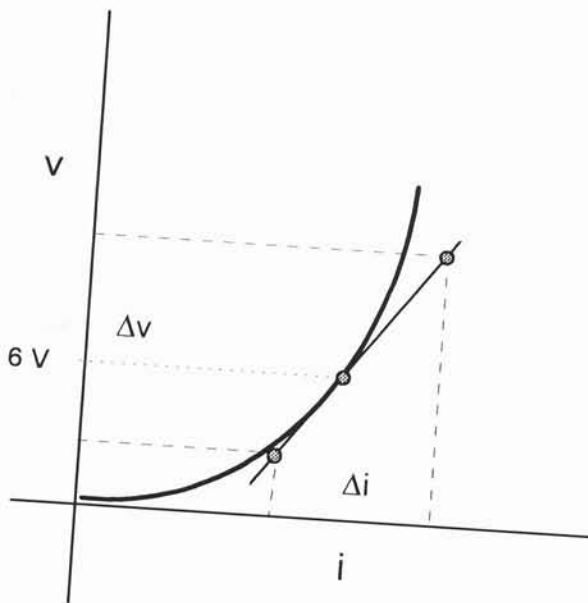
Potential required to make change in flow

Dead-time Delay - use in high frequency transmission lines and satellite communications

$$t_d = \frac{D}{v_p}$$

Where:  $D = \text{distance (m)}$   
 $v_p = \text{velocity of propagation (m/s)}$

Example: Non-linear resistance Volt-amp characteristic. Dynamic resistance.



Find R at 6 V operation

$$R = \frac{\Delta v}{\Delta i}$$

$$i_1 = 0.5 \quad i_2 = 0.504$$

$$v_1 = 5.95 \quad v_2 = 6.05$$

$$\Delta v = v_2 - v_1 \quad \Delta i = i_2 - i_1$$

$$\Delta v = 0.1$$

$$\Delta i = 0.004$$

$$R = \frac{\Delta v}{\Delta i}$$

$$R = 25 \text{ ohm}$$

Example: Sine voltage across a capacitor. What is the current?

$$e(t) = V_{\max} \cdot \sin(\omega \cdot t)$$

$$i(t) = C \cdot \frac{dv}{dt} = C \cdot \frac{d}{dt} V_{\max} \cdot \sin(\omega \cdot t) = C \cdot V_{\max} \cdot \omega \cdot \cos(\omega \cdot t)$$

90 degree lead between current and voltage

Example: Current pulse of 0.1 sec and amplitude of 0.1 mA applied to a capacitor produces a rise in voltage from 0 to 25 V. What is the capacitance?

$$i = C \cdot \frac{\Delta e}{\Delta t} \quad \text{solve for } C \quad i \cdot \frac{\Delta t}{\Delta e} = C$$

$$i = 0.1 \cdot 10^{-3} \cdot \text{amp} \quad e_1 = 0 \cdot \text{volt} \quad e_2 = 25 \cdot \text{volt}$$

$$t_1 = 0 \cdot \text{sec} \quad t_2 = .1 \cdot \text{sec}$$

$$\Delta e = e_2 - e_1 \quad \Delta t = t_2 - t_1$$

$$\Delta e = 25 \cdot \text{volt} \quad \Delta t = 0.1 \cdot \text{sec}$$

$$C = i \cdot \frac{\Delta t}{\Delta e} \quad C = 4 \cdot 10^{-7} \cdot \text{farad}$$

$$C = 4.0 \mu\text{F} \quad \text{Ans}$$

Example: voltage pulse of amplitude 5 with a duration of 0.02 sec is applied across an inductor. This causes a current increase from 1 amp to 2.1 amp. Find L.

$$e = L \cdot \frac{\Delta i}{\Delta t} \quad \text{Solve for inductance} \quad L = e \cdot \frac{\Delta t}{\Delta i}$$

$$t_1 = 0 \cdot \text{sec} \quad t_2 = 0.02 \cdot \text{sec} \quad e = 5 \cdot \text{volt}$$

$$i_1 = 1 \cdot \text{amp} \quad i_2 = 2.1 \cdot \text{amp}$$

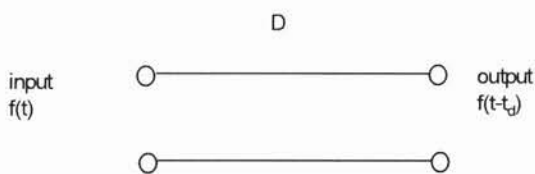
$$\Delta t = t_2 - t_1 \quad \Delta i = i_2 - i_1$$

$$\Delta t = 0.02 \cdot \text{sec} \quad \Delta i = 1.1 \cdot \text{amp}$$

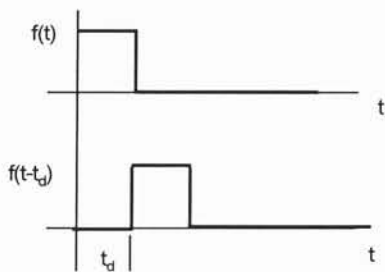
$$L = e \cdot \frac{\Delta t}{\Delta i} \quad L = 0.091 \cdot \text{henry} \quad (\text{V} \cdot \text{s} / \text{A})$$

Example: Electrical delays

Transmission lines



$$t_d = \frac{D}{v_p}$$



$v_p$  = velocity of propagation  
 typical values between  
 $2-3 \times 10^8$  m/s

a.) Find the delay of a 600 m transmission line with  
 $v_p = 2.3 \times 10^8$  m/s

b.) Find the delay of a satellite transmission with a path  
length of 2000 km and a velocity of propagation of  
 $3 \times 10^8$  m/s.

a.)

$$D = 600 \cdot \text{m} \quad v_p = 2.3 \cdot 10^8 \cdot \frac{\text{m}}{\text{sec}}$$

$$t_d = \frac{D}{v_p} \quad t_d = 2.609 \cdot 10^{-6} \cdot \text{sec}$$

$$t_d = 2.609 \mu\text{S} \quad \underline{\text{Ans}}$$

b.)

Convert km to m

$$2000 \cdot \text{km} \cdot 1000 \cdot \frac{\text{m}}{\text{km}} = 2 \cdot 10^6 \cdot \text{m}$$

$$t_d = \frac{2 \cdot 10^6 \cdot \text{m}}{3 \cdot 10^8 \cdot \text{sec}} \quad t_d = 6.667 \cdot 10^{-3} \cdot \frac{\text{m}}{\text{sec}}$$

$$t_d = 6.667 \text{ mS} \quad \underline{\text{Ans}}$$



## Liquid Flow Characteristics

Flow = volume of liquid/unit time  
(e.g. gallons/minute)

Potential = pressure

Liquid flow resistance - depends on the type of flow

### Types of liquid flow

Laminar - low velocity flows. Stream lines are parallel. Liquid flows in layers.

Linear flow resistance

turbulent - relatively high velocity flow.

non-linear flow resistance

Flow type determined by the Reynold's Number

$$R = \frac{\rho \cdot v \cdot d}{\mu}$$

Where  $\rho$  = density of the fluid (kg/m<sup>3</sup>)

$v$  = average velocity of the fluid (m/s)

$d$  = diameter of pipe (m)

$\mu$  = absolute viscosity of fluid (Pa-s)

Note: Reynold's number is dimensionless

Laminar flow :  $R < 2000$

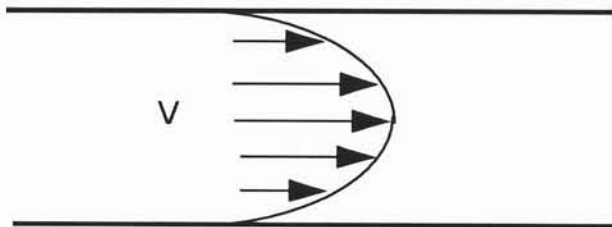
Turbulent flow:  $R > 4000$

Transition flow:  $2000 < R < 4000$

## Average velocity of liquid

$$v = \frac{Q}{A}$$

Laminar Flow



Velocity distribution

Where:

A = area of pipe (m<sup>2</sup>)

Q = flow (m<sup>3</sup>/s)

Since pipe diameter is usually given

$$v = \frac{4 \cdot Q}{\pi \cdot d^2}$$

Where:

Q = flow (m<sup>3</sup>/s)  
d = pipe diameter (m)

## Laminar Flow Equations for Round Pipes

$$p = R_L \cdot Q \quad (\text{Pa})$$

$$R_L = \frac{128 \cdot \mu \cdot l}{\pi \cdot d^4} \quad (\text{Pa} \cdot \text{s} / \text{m}^3)$$

Where:  $p$  = pressure drop (Pascals)

$R_L$  = laminar flow resistance

$Q$  = flow ( $m^3/s$ )

$l$  = length of pipe (m)

$\mu$  = absolute viscosity (Pa-s)

$d$  = pipe diameter (m)

### Turbulent flow Equations for Round Pipes

$$p = K_t \cdot Q^2 \quad \text{Pa}$$

$$R_t = 2 \cdot K_t \cdot Q \quad \text{Pa-s/m}^3$$

$$K_t = \frac{8 \cdot \rho \cdot f \cdot l}{\pi^2 \cdot d^5}$$

Where:  $f$  = friction factor (see table 3.3 p. 80 text)

$l$  = length (m)

$d$  = pipe diameter (m)

$\rho$  = density of liquid ( $kg/m^3$ )

$R_t$  = turbulent flow resistance ( $Pa-s/m^3$ )

$p$  = pressure (Pa)

$Q$  = flow ( $m^3/s$ )

Liquid Flow Capacitance - increase in volume of liquid required to make unit increase in pressure

$$C_L = \frac{\Delta V}{\Delta p}$$

Where:  $C_L$  = capacitance ( $\text{m}^3/\text{Pa}$ )

$\Delta V$  = volume change ( $\text{m}^3$ )

$\Delta p$  = pressure change (Pa)

Pressure relationship

$$\Delta p = \rho \cdot g \cdot \Delta H$$

Where:  $\rho$  = density of fluid

$g$  = acceleration due to gravity

$H$  = height of liquid in tank

Derive relationship

$$\begin{aligned} \Delta H &= \frac{\Delta V}{A} & \Delta p &= \rho \cdot g \cdot \frac{\Delta V}{A} \\ C_L &= \frac{\Delta V}{\Delta p} = \frac{\Delta V}{\rho \cdot g \cdot \frac{\Delta V}{A}} = \frac{A}{\rho \cdot g} \quad (\text{m}^3/\text{Pa}) \end{aligned}$$

## Liquid Flow Inertance (Inductance)

Amount of pressure drop required to increase flow rate by 1 unit/second

$$I_L = \frac{p}{\left( \frac{\Delta Q}{\Delta t} \right)}$$

Where:

$I_L$  = inertance (Pa/(m<sup>3</sup>/s<sup>2</sup>))

$p$  = pressure drop (Pa)

$\Delta Q/\Delta t$  = change in flow

Inertance defined using physical parameter

$$I_L = \frac{\rho \cdot l}{A} \quad (\text{Pa}/(\text{m}^3/\text{s}^2))$$

Where  $A$  = area of pipe

$\rho$  = density of liquid

$l$  = length of pipe

Dead-time Delay of Liquid - time required to transport liquid from one point to another

$$t_d = \frac{D}{v}$$

$v$  = average velocity of fluid (m/s)

$D$  = distance traveled (m)

Example 1: Oil at a temperature of 15 C flows in a horizontal, 1 cm diameter tube with a flow rate of 9.42 L/min. Tube length is 10 m. Find:

- Reynold's Number
- Flow resistance
- Pressure drop in tube

From Appendix A in text

$$\rho = 880 \cdot \frac{\text{kg}}{\text{m}^3} \quad \mu = 0.160 \cdot \text{Pa} \cdot \text{sec}$$

CONVERT ALL UNITS TO SI UNITS  $d = 1 \text{ cm} = 0.01 \text{ m}$

$$Q = (9.42 \text{ L/min}) (1.6667 \times 10^{-5} \text{ (m}^3/\text{s)/L/min}) \quad \text{Conversion in Appendix}$$

$$Q = 1.57 \times 10^{-4} \text{ m}^3/\text{s}$$

AVE. VELOCITY  $V = \frac{4Q}{\pi d^2} = \frac{4(1.57 \times 10^{-4} \text{ m}^3/\text{s})}{\pi (0.01 \text{ m})^2} = 2.00 \text{ m/s}$

Reynold's #  $R = \frac{\rho V d}{\mu} = \frac{(880 \text{ kg/m}^3)(2 \text{ m/s})(0.01 \text{ m})}{(0.160 \text{ Pa} \cdot \text{s})} = \boxed{109.95} \text{ ANS}$

$R < 2000$  LAMINAR FLOW LINEAR RELATIONSHIP BETWEEN FLOW AND PRESSURE

$$R_L = \frac{128 \mu L}{\pi d^4} = \frac{128(0.160 \text{ Pa} \cdot \text{s})(10 \text{ m})}{\pi (0.01 \text{ m})^4} = \boxed{6.519 \times 10^9 \text{ Pa} \cdot \text{s}/\text{m}^3} \text{ ANS}$$

Pressure Drop

$$p = R_L Q = (6.519 \times 10^9 \text{ Pa} \cdot \text{s}/\text{m}^3)(1.57 \times 10^{-4} \text{ m}^3/\text{s})$$

$$p = \boxed{1.0235 \times 10^6 \text{ Pa}} \text{ ANS}$$

Convert to PSI

$$p = (1.0235 \times 10^6 \text{ Pa})(1.45 \times 10^{-4} \text{ PSI/Pa}) = \boxed{148.41 \text{ PSI}} \text{ ANS}$$

Example 2: Water at 15 C flows through a commercial steel pipe with a diameter of 0.4 inch with a flow rate of 6 gal/min. The line is 50 ft long. Find:

Reynold's Number

Resistance to flow

Pressure drop in pipe

FROM APPENDIX A LIQUID DENSITY AND VISCOSITY

$$\rho = 1000 \text{ kg/m}^3 \quad \mu = 0.001 \text{ Pa}\cdot\text{s}$$

Convert English units to SI units

$$d = (0.4 \text{ in})(0.0254 \text{ m/in}) = 0.01016 \text{ m}$$

$$Q = (6 \text{ gal/min})(6.3088 \times 10^{-5} \text{ m}^3/\text{s/gpm}) = 3.7853 \times 10^{-4} \text{ m}^3/\text{s}$$

$$l = (50 \text{ ft})(0.3048 \text{ m/ft}) = 15.240 \text{ m}$$

Find Average Velocity  $V = \frac{4Q}{\pi d^2} = \frac{4(3.7853 \times 10^{-4} \text{ m}^3/\text{s})}{\pi(0.01016 \text{ m})^2}$

$$V = 4.669 \text{ m/s}$$

$$R = \frac{\rho V d}{\mu} = \frac{(1000 \text{ kg/m}^3)(4.669 \text{ m/s})(0.01016 \text{ m})}{0.001 \text{ Pa}\cdot\text{s}}$$

$$R = 47,490 \quad R > 4000$$

FLOW IS TURBULENT. MUST USE FRICTION FACTOR TABLE TO FIND RESISTANCE AND PRESSURE DROP

$$P = K_t Q^2 \quad R_t = 2 K_t Q \quad K_t = \frac{8 \rho f l}{\pi^2 d^5}$$

Find  $f$  from Table 3.3 p.80

Commercial steel pipe diameter 1-2 cm

$R$  between  $R_a > 10,000$   $R_b < 100,000$

$f$  values @  $R_a$   $f_a = 0.035$   $f_b = 0.028$

## Example 2 (cont.)

MUST USE LINEAR INTERPOLATION FORMULA  
TO FIND  $f$  for  $R = 47,440$

$$f = f_a + (f_b - f_a) \left[ \frac{R - R_a}{R_b - R_a} \right]$$

$f$  must be between  
 $f_a$  and  $f_b$

$$f = 0.035 + (0.028 - 0.035) \left[ \frac{47440 - 10,000}{100,000 - 10,000} \right]$$

$$f = 0.035 + (-0.007) \left( \frac{3.744}{9} \right)$$

$$f = 0.03209$$

Now find  $K_t$

$$K_t = \frac{8(1000 \text{ kg/m}^3)(0.03209)(15.24 \text{ m})}{\pi^2 (0.01016 \text{ m})^5}$$

$$K_t = 3.6614 \times 10^{12} \text{ Pa-s/m}^6$$

Find  $R_t$

$$R_t = 2K_t Q = 2(3.6614 \times 10^{12} \text{ Pa-s/m}^6)(3.7853 \times 10^{-4} \text{ m}^3/\text{s})$$

$$R_t = 2.7719 \times 10^9 \text{ Pa-s/m}^3 \quad \underline{\underline{\text{ANS}}}$$

CONVERT TO ENGLISH UNITS

$$(2.7719 \times 10^9 \text{ Pa-s/m}^3)(9.148 \times 10^{-9}) = 25.357 \text{ psi/gpm} \quad \underline{\underline{\text{ANS}}}$$

↳ from appendix

$$P = K_t Q^2$$

$$P = (3.6614 \times 10^{12} \text{ Pa-s/m}^6)(3.7853 \times 10^{-4} \text{ m}^3/\text{s})^2$$

$$P = 5.246 \times 10^5 \text{ Pa}$$

CONVERT TO ENGLISH UNITS

$$P = (5.246 \times 10^5 \text{ Pa})(1.45 \times 10^{-4} \text{ psi/Pa}) = 76.07 \text{ psi}$$

ANS



Example 4: A tank has a diameter of 1.83 meters and a height of 10 ft. Determine the capacitance of the tank when it holds: a.) water b.) oil c.) kerosene d.) gasoline

$$a.) C_L = \frac{A}{\rho g} \quad A = \frac{\pi d^2}{4} = \frac{\pi (1.83 \text{ m})^2}{4} = 2.63 \text{ m}^2$$

$$\rho = 1000 \text{ kg/m}^3 \quad C_L = \frac{2.63 \text{ m}^2}{1000 \text{ kg/m}^3 \cdot 9.81 \text{ m/s}^2} = 2.68 \times 10^{-4} \text{ m}^3/\text{Pa}$$

$$b.) \rho = 880 \text{ kg/m}^3 \quad C_L = \frac{2.63 \text{ m}^2}{(880 \text{ kg/m}^3)(9.81 \text{ m/s}^2)} = 3.05 \times 10^{-4} \text{ m}^3/\text{Pa}$$

$$c.) \rho = 800 \text{ kg/m}^3 \quad C_L = \frac{2.63 \text{ m}^2}{800 \text{ kg/m}^3 (9.81 \text{ m/s}^2)} = 3.35 \times 10^{-4} \text{ m}^3/\text{Pa}$$

$$d.) \rho = 740 \text{ kg/m}^3 \quad C_L = \frac{2.63 \text{ m}^2}{740 \text{ kg/m}^3 (9.81 \text{ m/s}^2)} = 3.62 \times 10^{-4} \text{ m}^3/\text{Pa}$$

Example 5: Determine the inertance of water in a pipe with a diameter of 2.1 cm and a length of 65 meters.

$$I_L = \frac{\rho l}{A} \quad \rho = 1000 \text{ kg/m}^3 \quad A = \frac{\pi d^2}{4}$$

$$l = 65 \text{ m}$$

$$A = \frac{\pi (0.021 \text{ m})^2}{4}$$

$$I_L = \frac{(1000 \text{ kg/m}^3)(65 \text{ m})}{3.46 \times 10^{-4} \text{ m}^2}$$

$$A = 3.46 \times 10^{-4} \text{ m}^2$$

$$I_L = 1.88 \times 10^8 \text{ Pa}/(\text{m}^3/\text{s}^2)$$

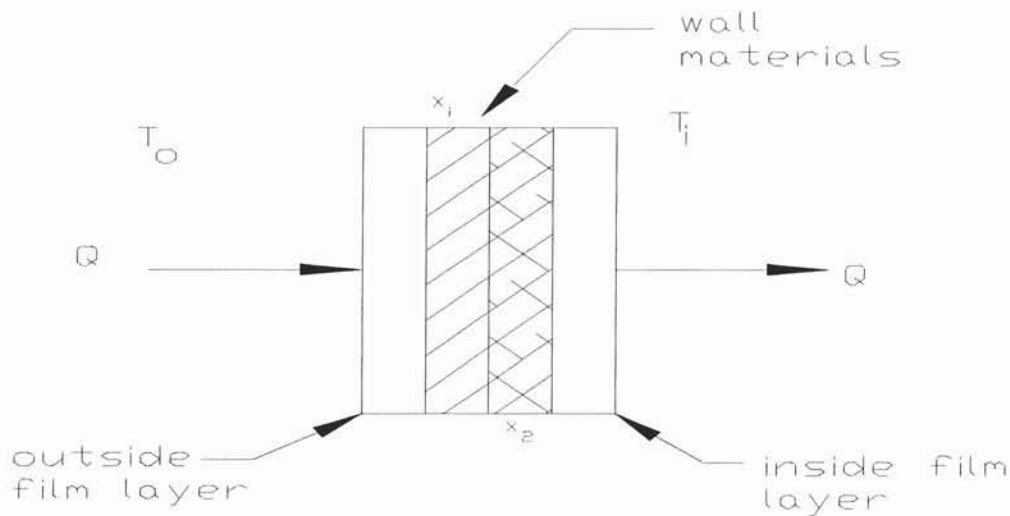
ANS  
5

## Thermal Elements

### Thermal Conduction and Convection

Thermal Resistance - opposition to heat flow. Heat flows from high temperature to low temperature

$$Q = \frac{T_o - T_i}{R_T} \quad \text{Watts}$$



$R_u =$  unit thermal resistance

$$R_T = \frac{R_u}{A} \quad \text{K/W} \quad A = \text{area of surface} \quad Q = \frac{(T_o - T_i) \cdot A}{R_u} \quad \text{Watts}$$

Resistance of film layers depends on type of fluid, velocity of fluid. High velocity flow makes thin film. Convection cooling. "Wind Chill".

Film Conductance  $h = \frac{1}{R_{\text{film}}}$  Called film coefficient

Film coefficients: Natural Convection In Air

a). Horizontal surface facing up

$$h = 2.5 \cdot T_d^{.25}$$

$T_d$  = temperature difference between wall and fluid

a). Horizontal surface facing down

$$h = 1.32 \cdot T_d^{.25}$$

a). vertical surfaces

$$h = 1.78 \cdot T_d^{.25}$$

Natural convection in still water

$$h = 2.26 \cdot (T_w + 34.3) \cdot T_d^{.5}$$

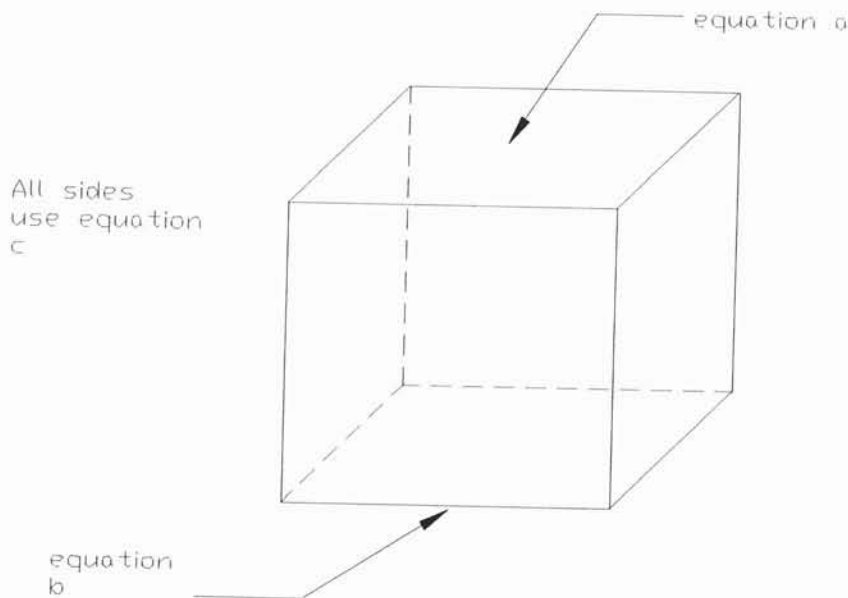
$T_w$  = water temperature

Forced convection-air against smooth surfaces and inside pipes

$$v_{\text{air}} \leq 4.6 \cdot \frac{\text{m}}{\text{s}} \quad h = 4.54 + 4.1 \cdot v_{\text{air}}$$

$$v_{\text{air}} > 4.6 \cdot \frac{\text{m}}{\text{s}} \quad h = 7.75 \cdot v_{\text{air}}^{0.75}$$

Using the natural convection in air formulas



Thermal resistance of inner solid layers given by

$$R_{ui} = \frac{x_i}{k_i} \quad \text{K-m}^2/\text{W}$$

Where:  $x_i$  = thickness of i-th material  
 $k_i$  = thermal conductivity of material  
(see appendix in text)

Unit resistance of a composite wall is the sum of inside and outside film resistances and the resistances comprising the wall.

Series resistance analogy  
 Temperature analogous to voltages



$$R_u = \frac{1}{h_o} + \frac{x_1}{k_1} + \frac{x_2}{k_2} + \dots + \frac{x_n}{k_n} + \frac{1}{h_i} \quad \text{K-m}^2/\text{W}$$

$$R_T = \frac{1}{A} \left( \frac{1}{h_o} + \frac{x_1}{k_1} + \frac{x_2}{k_2} + \dots + \frac{x_n}{k_n} + \frac{1}{h_i} \right) \quad \text{K/W}$$

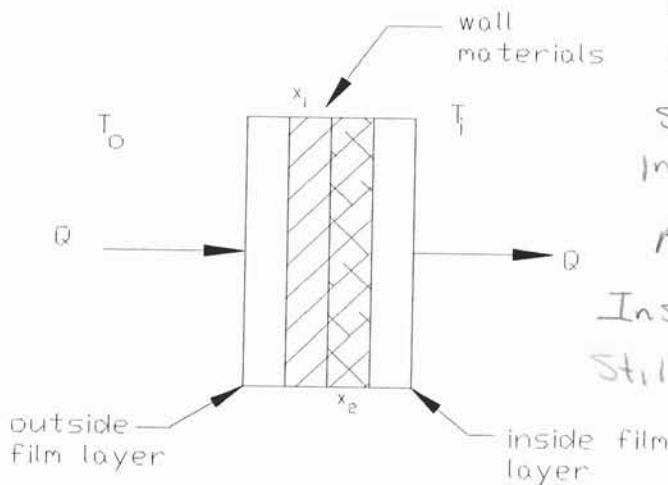
Thermal Capacitance - increase in heat required to make a unit change in temperature (SI unit J/K)

Heat Capacity (specific heat) - heat required to raise temperature of 1 kg of material by 1 K.

$$C_T = m \cdot S_T$$

Where:  $C_T$  = thermal capacitance  
 $m$  = mass (kg)  
 $S_T$  = heat capacity (J/kg)

Example 1: A wall section shown below has 2 layers: steel 1 cm thick, and insulation 2 cm thick. Still Water at 45 C is inside the wall. The temperature difference between the water and the inner wall is estimated to be 10 C. The outer wall is surrounded by air at a temperature of 85 C that has a velocity of 6 m/s. The wall dimensions are 2 m x 3 m. Determine the unit thermal resistance, the total thermal resistance, the total heat flow, the direction of heat flow.



Find  $K_1$  and  $K_2$  from Appendix A

steel  $K_1 = 45 \text{ W/m}\cdot\text{K}$

Insulation  $K_2 = 0.036 \text{ W/m}\cdot\text{K}$

$A = 2 \text{ m} \times 3 \text{ m} = 6 \text{ m}^2$

Inside film coefficient

Still water

$$h_i = 2.26(T_w + 34.3)T_d^{0.5}$$

$$T_w = 45 \text{ C} \quad T_d = 10 \text{ C}$$

$$h_i = 2.26(45 + 34.3) = \frac{566.7 \text{ W/m}^2\cdot\text{K}}{2}$$

Outside film coefficient  
air forced convection

$$V_{\text{air}} = 6 \text{ m/s} > 4.6 \text{ m/s}$$

$$\text{use } h_o = 7.75 V_{\text{air}}^{0.75} = 7.75 (6 \text{ m/s})^{0.75} = 29.71 \text{ W/m}^2\cdot\text{K}$$

$$R_u = \frac{1}{h_o} + \frac{x_1}{K_1} + \frac{x_2}{K_2} + \frac{1}{h_i} = \frac{1}{566.7 \text{ W/m}^2\cdot\text{K}} + \frac{0.01 \text{ m}}{45 \text{ W/m}\cdot\text{K}} + \frac{0.02 \text{ m}}{0.036 \text{ W/m}\cdot\text{K}} + \frac{1}{29.71 \text{ W/m}^2\cdot\text{K}}$$

$$X_1 = 0.01 \text{ m} \quad R_u = 0.001765 + 0.000222 + 0.5556 + 0.033656$$

$$X_2 = 0.02 \text{ m} \quad R_u = 0.5912 \text{ K}\cdot\text{m}^2/\text{W} \quad \underline{\text{ANS}}$$

$$R_T = \frac{R_u}{A} = \frac{0.5912 \text{ K}\cdot\text{m}^2/\text{W}}{6 \text{ m}^2} = 0.09854 \text{ K/W} \quad \underline{\text{ANS}}$$

Into Wall

$$\text{Heat Flow } Q = \frac{T_o - T_i}{R_T} = \frac{(85 - 45) \text{ C}}{0.09854 \text{ K/W}} = 406 \text{ W} \quad \underline{\underline{\text{ANS}}}$$

Example 2: Determine the thermal resistance for each of the following film conditions: a.) Natural convection in still air of vertical surface  $T_d = 20$  C. b.) Natural convection in still water where  $T_d = 30$  C and  $T_w$  is 20 C c.) Forced convection in air with velocity of 4 m/s.

a.) Natural Convection air on vertical surface

$$T_d = 20\text{C} \quad h = 1.78 T_d^{0.25} = 1.78 (20)^{0.25}$$

$$h = 3.764 \text{ W/m}^2\text{-K}$$

$$R_u = \frac{1}{h} = \boxed{0.266 \text{ K-m}^2/\text{W}} \quad \frac{\text{ANS}}{3}$$

b.) Natural Convection in still water

$$T_d = 30 \quad T_w = 20\text{C} \quad h = 2.26 (T_w + 34.3) T_d^{0.5}$$

$$h = 2.26 (20 + 34.3) (30)^{0.5}$$

$$h = 2.26 (54.3) (30)^{0.5}$$

$$h = 672.2 \text{ W/m}^2\text{-K}$$

$$R_u = \frac{1}{h} = \boxed{0.00149 \text{ K-m}^2/\text{W}} \quad \frac{\text{ANS}}{3}$$

c.) Forced Convection Air  $V_{\text{air}} \leq 4.6 \text{ m/s}$

$$V_{\text{air}} = 4 \text{ m/s} \quad h = 4.54 + 4.1 V_{\text{air}}$$

$$h = 4.54 + 4.1 (4) = 20.94 \text{ W/m}^2\text{-K}$$

$$R_u = \frac{1}{h} = \boxed{0.0478 \text{ K-m}^2/\text{W}} \quad \frac{\text{ANS}}{3}$$

## Mechanical Elements

Mechanical resistance = friction

Opposition to motion. Force required to increase velocity.

Viscous friction - friction of motion. Related to velocity

Coulomb friction - force independent of velocity

Example of viscous friction - shock absorber, dash pot

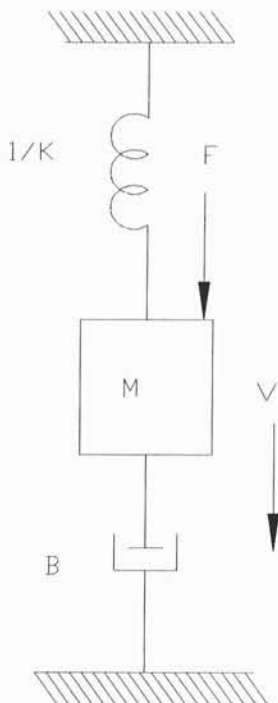
linear viscous friction

$$R_m = B = \frac{F}{v} \quad \text{N-s/m}$$

Non-linear viscous friction

$$B = \frac{\Delta F}{\Delta v} \quad B = \frac{dF}{dt}$$

At operating point use tangent line to find approx. B



Mechanical Capacitance - change in length of spring required to make unit increase in force. Inverse spring constant

$$C_m = \frac{1}{K} \quad \text{N/m}$$

Mechanical inertia (mass) - force required to make unit increase in acceleration

$$F_{ave} = m \cdot \frac{\Delta v}{\Delta t} \quad F = m \cdot \frac{dv}{dt}$$

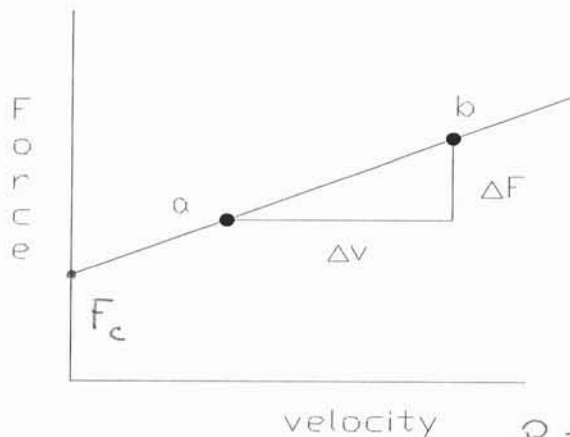


Mechanical Dead-time - time required to transport material from one place to another

$$t_d = \frac{D}{v}$$

Where  $D$  = distance  
 $v$  = velocity of material

Example 1: A mechanical system consists of a sliding load and a shock absorber. The force versus velocity curve is shown below



Test data

Run	F (N)	v (m/s)
a	7.1	10.5
b	9.6	15.75

Determine  $R_m$  and coulomb friction

$B = R_m = \frac{\Delta F}{\Delta v}$  use point slope form of line

USE  $F = F_c + Bv$

$$F - F_1 = \frac{\Delta F}{\Delta v} (v - v_1) \Rightarrow F - 7.1 = \frac{9.6 - 7.1}{15.75 - 10.5} (v - 10.5)$$

$$F - 7.1 = \frac{2.5}{5.25} (v - 10.5)$$

$$F - 7.1 = 0.4762v - 5$$

$$F = 0.4762v + 2.1 \text{ N}$$

$$B = 0.4762 \text{ N-s/m}$$

$$F_c = 2.1 \text{ N}$$

ANS  
 $\underline{\underline{\quad}}$

# Modeling Physical Systems

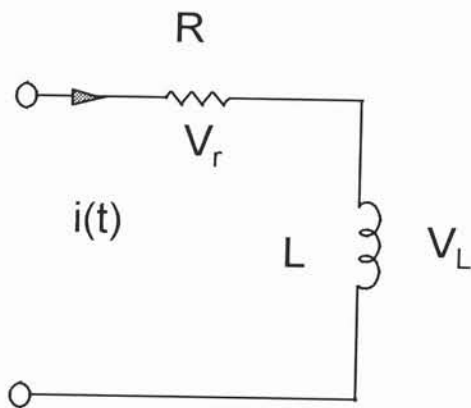
**Linear systems** - modeled with linear differential equations

Derivatives capture how system variables change with time

## Definition

Linear Differential Equation - a linear combination of derivatives of an unknown function and the unknown function

**Example** Series RL circuit. What does current do over time



$$L \cdot \frac{d}{dt} i(t) + R \cdot i(t) = 0$$

To solve - find how  $i$  varies with time- $i(t)$

Can do analytically or numerically

## More Complex Differential Equations

Can have higher order derivatives... 2, 3, 4...etc.

$$\frac{d^2}{dt^2} i(t) + 2 \cdot \frac{d}{dt} i(t) - 7 \cdot i(t) = 0$$

2nd order- linear ODE (ordinary differential equation)

$$\frac{d^2}{dt^2} i - 3 \cdot \frac{d}{dt} i + i^2 = 0 \quad i = i(t)$$

2nd order non-linear

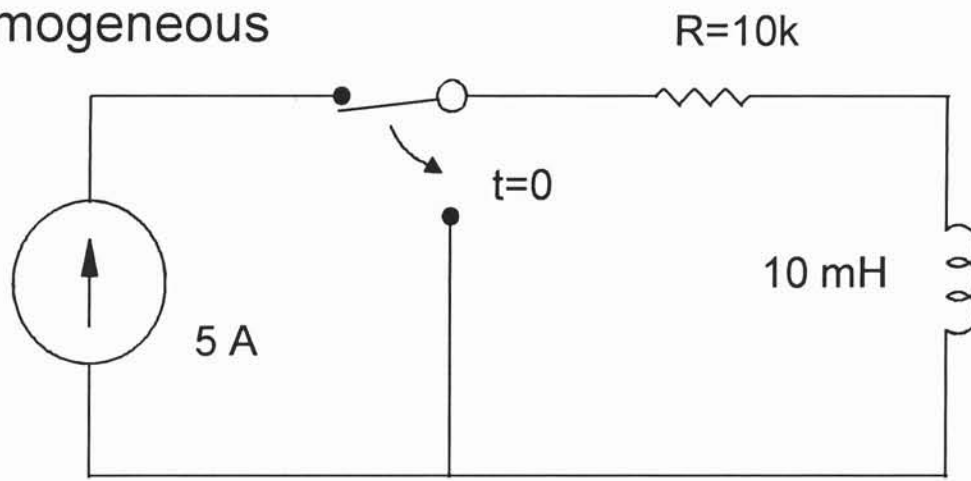
$$\left( \frac{d^2}{dt^2} v \right) \cdot \sin(v) + v = 0$$

2nd order non-linear

When right-hand side (RHS) is 0, equation called homogeneous. Implies no outside stimulation.

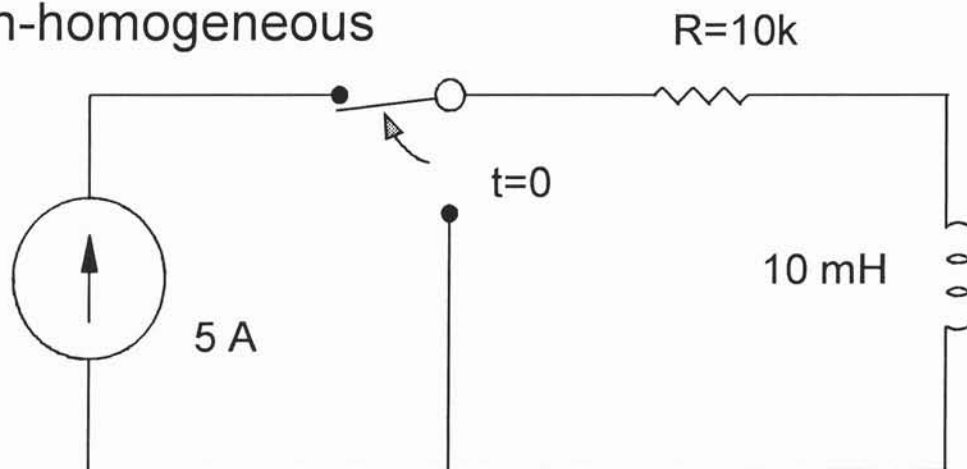
# Examples

homogeneous



No current source after the switch is thrown.

non-homogeneous



Energize relay with the current source  
Current source drives the system.

# Differential equations for control

Equations have constant coefficients and are linear.

Single input stimulation  $r(t)$

Single output variable  $x(t)$

$$a_n \cdot \frac{d^n}{dt^n} x(t) + \dots + a_2 \cdot \frac{d^2}{dt^2} x(t) + a_1 \cdot \frac{d}{dt} x(t) + a_0 \cdot x(t) = b_0 \cdot r(t)$$

$a_n, \dots, a_2, a_1, a_0$  and  $b_0$  are constants

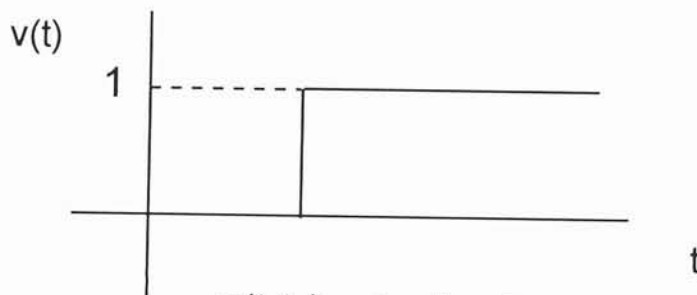
What can  $r(t)$  be?

constant -5, 3, 10

ramp 3t, -10t, 0.2t-3

sin/cos 100sin( $\omega t$ ), -5cos( $\omega t + \theta$ )  $w$  and  $q$  are constants

Unit-step (square wave)



$u(t-t_d) = 1$  after  $t_d$   
0 before

## Examples

Identify which of the following equations are linear and which are non-homogeneous

$$-3 \cdot \frac{d}{dt} x(t) + x(t)^2 = 0$$

$$\left( \frac{d}{dt} v \right)^2 + \sin(v) = 0$$

$$6 \cdot \frac{d}{dt} v + 2 \cdot v = V_m \cdot \sin(\omega t)$$

$$4 \cdot \frac{d^2}{dt^2} i - 2 \cdot \frac{d}{dt} i - 7 \cdot i - I_0 \cdot e^{-\frac{t}{\tau}} = 0$$

$$L \cdot \frac{d}{dt} i(t) + R \cdot i(t) = 0$$

# Characteristics of Linear Systems

1.) Multiplying by constant is reflected through system.

If input  $r(t)$  gives output  $x(t)$  then,  
 $K(r(t))$  gives  $K(x(t))$   
I/O proportional

2.) Superposition from circuits holds

If input  $r_1(t)$  gives  $y_1(t)$  and  
input  $r_2(t)$  gives  $y_2(t)$  then  
total output to inputs is

$$r_1(t) + r_2(t) = y_1(t) + y_2(t)$$

Total output is the sum of the individual input responses.

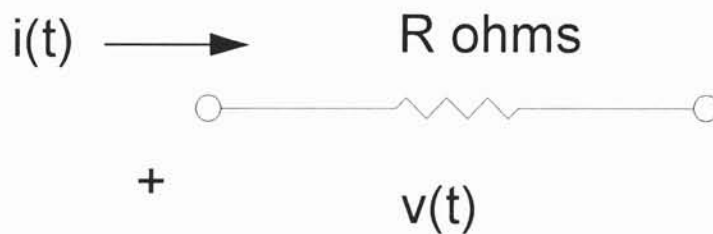
From circuits - transients and sine steady-state

# Deriving Dynamic Equations

## Input/output relationships

Dynamics represented by integrals and derivatives

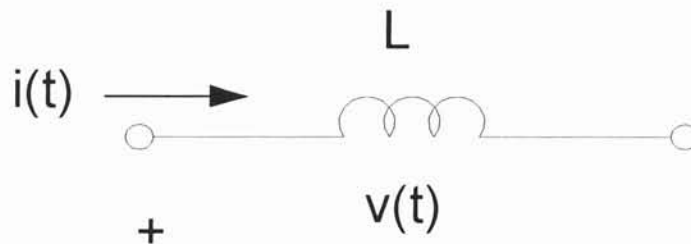
Electrical elements: Resistance



Defining equations  $v(t) = R \cdot i(t)$

$$i(t) = G \cdot v(t) \quad \frac{1}{R} = G$$

Inductance (henrys)



Defining equations

$$v(t) = L \cdot \frac{d}{dt} i(t)$$

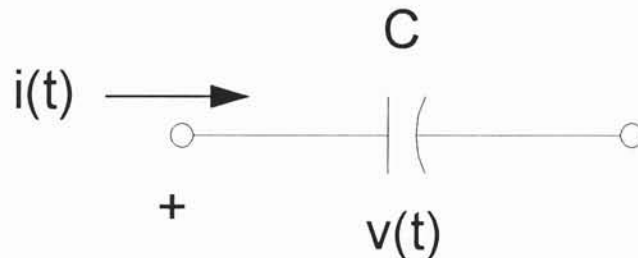
$$i(t) = \frac{1}{L} \int_0^t v(t) dt + i(0)$$

initial current  
at start of time



## Lumped Circuit Elements: capacitance

Capacitance (farads)



initial voltage at  
start time

Defining equations

$$i(t) = C \cdot \frac{d}{dt} v(t) \quad v(t) = \frac{1}{C} \int_0^t i(t) dt + v(0)$$

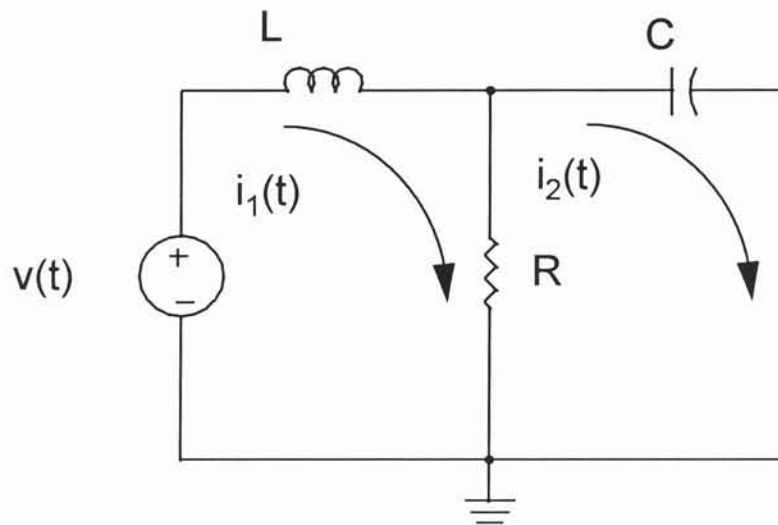
An arrow points from the text "initial voltage at start time" to the  $v(0)$  term in the second equation.

All law from circuit theory hold for the analysis of lumped circuits. KCL, KVL, mesh analysis, nodal analysis can all be performed. Substitute the appropriate integral or derivative into the mesh or nodal formulation.

When the current or voltage in a circuit element involves the two currents or voltages, the derivative or integral take the difference of the voltages or currents

## Example

Write mesh equations for the circuit below using the lumped circuit element representations



KVL around mesh 1

$$-v(t) + L \cdot \frac{d}{dt} i_1(t) + R_1(i_1(t) - i_2(t)) = 0$$

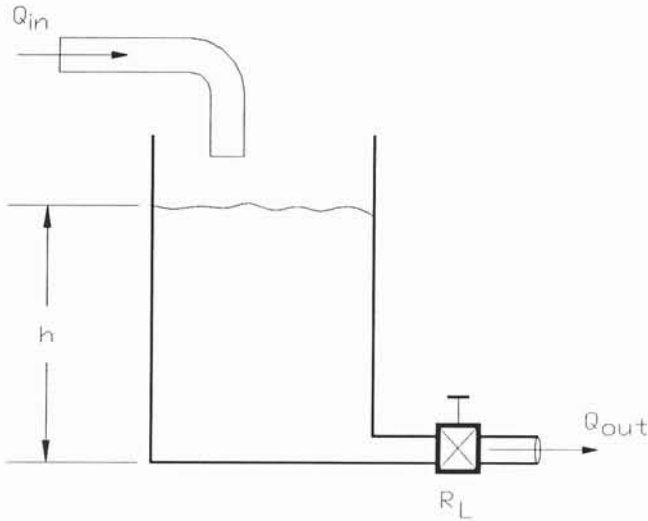
$$L \cdot \frac{d}{dt} i_1(t) + R_1(i_1(t) - i_2(t)) = v(t)$$

KVL around mesh 2

$$R_1(i_2(t) - i_1(t)) + \frac{1}{C} \int_0^t i(t) dt + v(0) + R_2 i_2(t) = 0$$

system of integral-differential equations in unknown functions  $i_1(t)$  and  $i_2(t)$

## Self-regulating tank system



Need relationship for how level changes with time.  
Derive differential equation

$$\Delta V = (Q_{in} - Q_{out}) \cdot \Delta t$$

level change

$$\Delta h = \frac{\Delta V}{A}$$

$$\Delta h = \frac{\Delta V}{A} = \frac{(Q_{in} - Q_{out}) \cdot \Delta t}{A}$$

average level change over time interval  $\Delta t$

$$\frac{\Delta h}{\Delta t} = \frac{(Q_{in} - Q_{out})}{A} \quad \text{take limit } \Delta t \rightarrow 0 \quad \frac{dh}{dt} = \frac{(Q_{in} - Q_{out})}{A}$$

Assume laminar flow for simplicity.  $Q_{out}$  determined by  $p$  of tank.

$$P = R_L \cdot Q_{out} \quad P = \rho \cdot g \cdot h$$

Combine these two equations and solve for  $Q_{out}$

$$\frac{P}{R_L} = Q_{out}$$

$$\frac{\rho \cdot g \cdot h}{R_L} = Q_{out}$$

## Self-regulating tank system (cont.)

$$\frac{dh}{dt} = \frac{\left( Q_{in} - \frac{\rho \cdot g \cdot h}{R_L} \right)}{A} \quad \longrightarrow \quad \frac{dh}{dt} = \left( \frac{Q_{in}}{A} - \frac{\rho \cdot g \cdot h}{R_L \cdot A} \right)$$

bring all terms with  
h or derivative of h  
to one side

$$\frac{dh}{dt} + \left( \frac{\rho \cdot g}{R_L \cdot A} \right) \cdot h = \frac{Q_{in}}{A}$$

multiply through by  
coefficient on h  
Remember finding h(t)

$$\left( \frac{R_L \cdot A}{\rho \cdot g} \right) \cdot \frac{dh}{dt} + h = \frac{R_L}{\rho \cdot g} \cdot Q_{in}$$

$Q_{in}$  is independent of the liquid height. consider it along with  
density, area and flow resistance to be constant

Let

$$\tau = \frac{R_L \cdot A}{\rho \cdot g} \quad G = \frac{R_L}{\rho \cdot g}$$

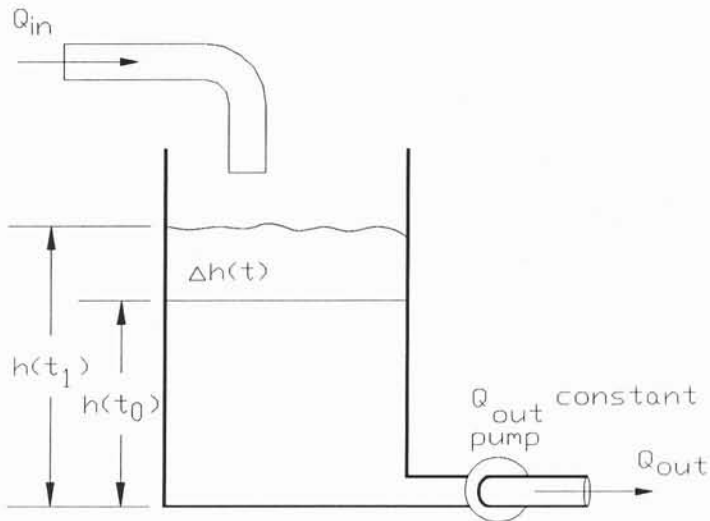
$$\tau \cdot \frac{dh}{dt} + h = G \cdot Q_{in}$$

non-homogeneous equation that  
determines how height of liquid in  
tank varies with time

Shut off value to find natural response of tank for some initial  
height

## Non-regulating Tank

Output flow is fixed by pump flow rate. It is independent of the liquid height in tank.



$$\Delta h = \frac{\Delta V}{A} = \frac{(Q_{in} - Q_{out}) \cdot \Delta t}{A}$$

Define

$$\Delta t = t_1 - t_0$$

$$\Delta h(t) = h(t_1) - h(t_0)$$

$$\frac{\Delta h}{\Delta t} = \frac{(Q_{in} - Q_{out})}{A}$$

take limit  $\Delta t \rightarrow 0$

$$\frac{dh}{dt} = \frac{(Q_{in} - Q_{out})}{A}$$

Right hand side is independent of  $h$ . To solve integrate both sides with respect to  $t$

$$\int_{t_0}^{t_1} \frac{dh}{dt} dt = \int_{t_0}^{t_1} \frac{1}{A} \cdot (Q_{in} - Q_{out}) dt$$

## Non-regulating Tank (cont.)

From Calculus

$$h(t_1) - h(t_0) = \int_{t_0}^{t_1} \frac{1}{A} \cdot (Q_{in} - Q_{out}) dt$$

$$\Delta h(t) = \int_{t_0}^{t_1} \frac{1}{A} \cdot (Q_{in} - Q_{out}) dt$$

Note: Right hand side is not a function of  $t$ . It is all a constant and can be taken out of the integral

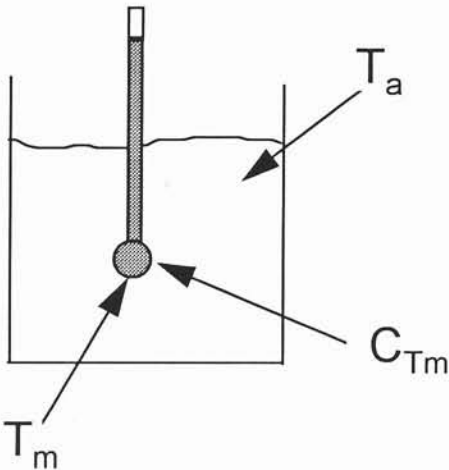
Change in height depends on the difference in flows and the time that the difference is applied.

If  $Q_{in}$  is not a constant but changes with time, the formula below models the response of the tank system

$$\Delta h(t) = \int_{t_0}^{t_1} \frac{1}{A} \cdot (Q_{in}(t) - Q_{out}) dt$$

Solution techniques: convert all equations into derivatives only. Approximate derivatives using mathematical methods calculate approximate derivative values for some small increment in time. Results are a list of computed points that approximate variable over a time interval. Graph these points to see system response

### Liquid Filled Thermometer



$T_a$  = fluid temperature  
 $T_m$  = measured temperature  
 $C_{Tm}$  = thermal capacitance of thermometer

How does measured temperature change with time. Heat transferred to thermometer depends on  $\Delta T$ ,  $R_T$  and time interval

$$\Delta Q = \frac{(T_a - T_m) \cdot \Delta t}{R_T}$$

definition of thermal capacitance

$$\frac{\Delta Q}{\Delta T_m} = C_{Tm}$$

$$\Delta T_m = \frac{\Delta Q}{C_{Tm}}$$

$$\frac{\Delta Q}{C_{Tm}} = \frac{(T_a - T_m) \cdot \Delta t}{R_T \cdot C_{Tm}}$$

$$\Delta T_m = \frac{(T_a - T_m) \cdot \Delta t}{R_T \cdot C_{Tm}}$$

## Liquid Filled Thermometer (cont.)

Determine the average change in temperature

$$\frac{\Delta T_m}{\Delta t} = \frac{(T_a - T_m)}{R_T \cdot C_{Tm}}$$

take limit  $\Delta t \rightarrow 0$

$$\frac{dT_m}{dt} = \frac{(T_a - T_m)}{R_T \cdot C_{Tm}}$$

Get all  $T_m$  and its derivatives on one side of the equation

$$\frac{dT_m}{dt} = \frac{T_a}{R_T \cdot C_{Tm}} - \frac{T_m}{R_T \cdot C_{Tm}}$$

$$\frac{dT_m}{dt} + \frac{T_m}{R_T \cdot C_{Tm}} = \frac{T_a}{R_T \cdot C_{Tm}}$$

$$R_T \cdot C_{Tm} \cdot \frac{dT_m}{dt} + T_m = T_a$$

This is similar in form to the self-regulating tank equation

This is a non-homogeneous differential equation that describes how the measured temperature changes with time



# Mechanical Systems

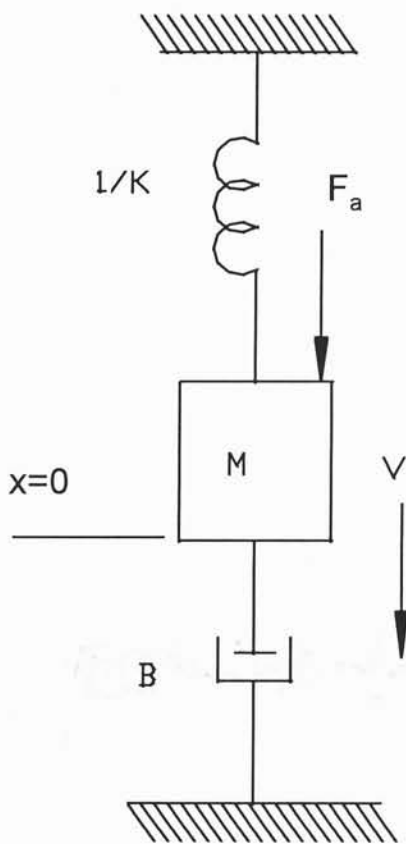
## Pneumatic Control Valve Position

Determine how the position of a air actuated control valve changes with time after pressure is applied

### Schematic of value

(See text p.116 for details)

$$F_a = P_a(A) = \text{input air force}$$



All forces must balance at each instance in time so:

$$F_a = F_I(t) + F_R(t) + F_C(t)$$

Where  $F_I(t)$  = inertial force

$F_R(t)$  = viscous friction force

$F_C(t)$  = spring force

Need equation that relates position, x to time

$$F_I = m \cdot a$$

$$a = \frac{d}{dt} v = \frac{d^2}{dt^2} x$$

$$v = \frac{d}{dt} x$$

Inertial force

$$F_I = m \cdot \frac{d^2}{dt^2} x$$

## Pneumatic Control Value Position, (cont.)

Inertial force

$$F_I = m \cdot \frac{d^2}{dt^2} x$$

Acceleration is second derivative of position,  $x$

Friction force

$$F_R = B \cdot v = B \cdot \left( \frac{d}{dt} x \right)$$

Viscous friction for laminar flow is proportional to the velocity. Velocity is the rate of change of position

Spring force

$$x = C_m \cdot F_C \quad F_C = \frac{x}{C_m} \quad \frac{1}{C_m} = k$$

$$F_C(t) = k \cdot x(t) = k \int x(t) dt$$

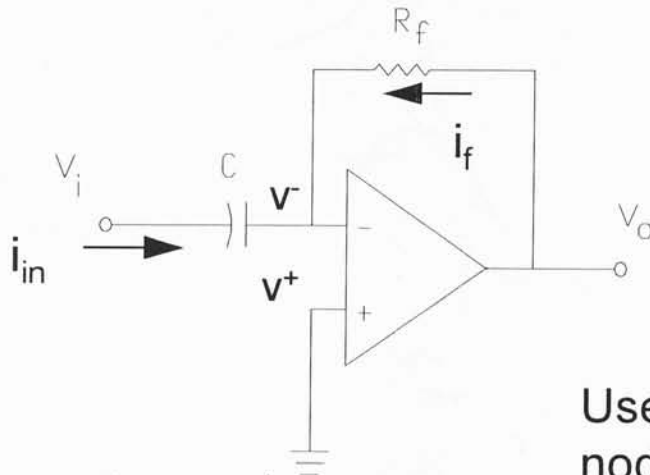
Force from spring is proportional to its length,  $x$ .  $C_m$  is spring capacitance,  $k$  = spring constant.

$$F_a = m \cdot \frac{d^2}{dt^2} x + B \cdot \left( \frac{d}{dt} x \right) + \frac{1}{C_m} \cdot x$$

$F_a$  is constant. Equation describes how position changes with time. Second order equation.

# Simulating differentiation with OP AMPS

Use rules of circuit analysis and ideal OP AMPS to find the input/output relationship for the circuit below.



Rules of OP AMPS

no current flows into OP AMP

$$V^- = V^+$$

Use nodal analysis at inverting node of OP AMP

Sum Currents

$$i_{in}(t) + i_f(t) = 0 \rightarrow i_{in}(t) = -i_f(t)$$

$$i_{in}(t) = i_c(t) \text{ Current in } C$$

define currents in terms of voltage

$$i_{in}(t) = C \frac{d}{dt} (V_{in}(t) - V^-(t))$$

$$i_f(t) = \frac{V_o(t) - V^-(t)}{R_f}$$

$$C \frac{d}{dt} (V_{in}(t) - V^-(t)) = \frac{-V_o(t) - V^-(t)}{R_f}$$

$$i_c(t) = C \frac{d}{dt} V_c(t)$$

$$V_c(t) = V_{in}(t) - V^-(t)$$

$$V^-(t) = V^+(t) = 0 \text{ Pos. terminal Grounded}$$

$$\rightarrow C \frac{d}{dt} V_{in}(t) = \frac{-V_o(t)}{R_f}$$

$$-R_f C \frac{d}{dt} V_{in}(t) = V_o(t)$$

write as input/output

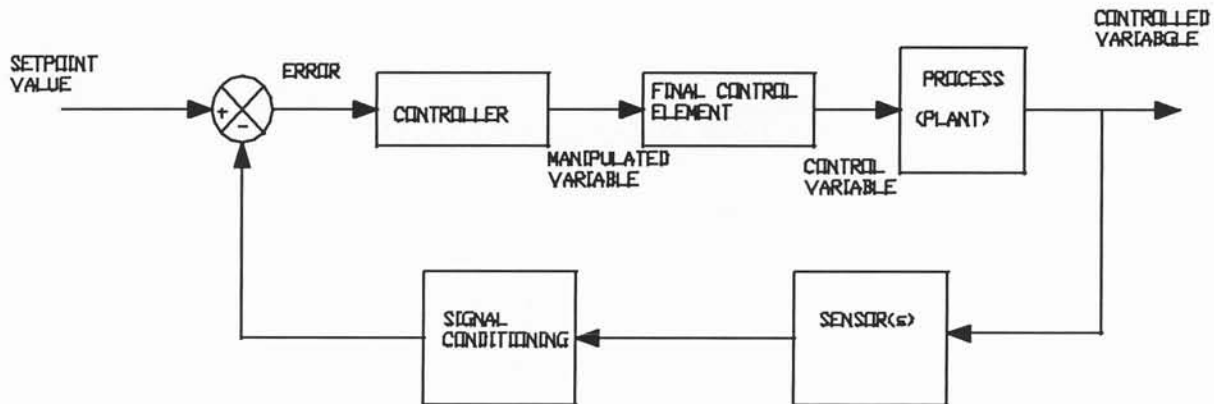
$$V_o(t) = -R_f C \frac{d}{dt} V_{in}(t)$$

output

input

# Proportional Control Action

## Basic control loop



Simplest form of controller - amplify error and apply signal to the process through final control element.

Example: Dc motor speed control

control variable = motor speed

sensor/signal conditioning = speed transducer

setpoint = desired speed

final control element = variable dc voltage supply

control variable = motor terminal voltage

manipulated variable = dc voltage control voltage

The control relationship for proportional control

Amplify error and send to final control element.

$$C_o = K_p e + C_b$$

Where  $C_o$  = the controller output

$C_b$  = the controller output with  $e = 0$

$K_p$  = the proportional gain

$e$  = the control error

$$e = SP - \text{Measurement},$$

$e$  = error signal

SP = setpoint value

Measurement = sensor measurement

$e$  can be positive or negative

$e$  used to take corrective action

Measurement > SP  $e$  negative - output decreases

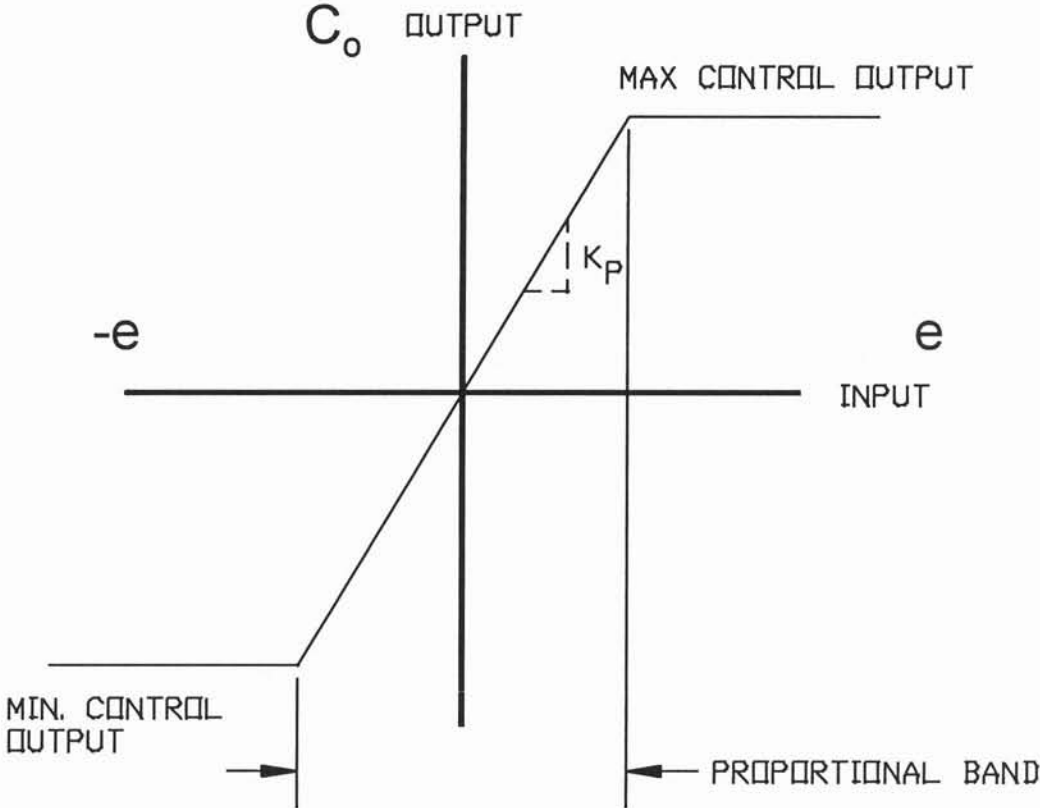
Measurement < SP  $e$  positive - output increases

Practical controller output devices have limits on  $C_o$  and final control element.

Examples: flow valve position

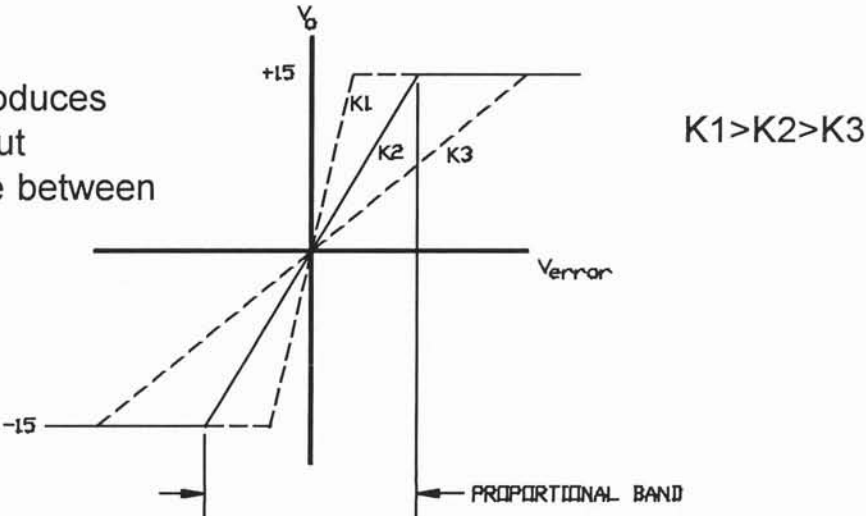
motor power supply

# Controller Gain Plots



Proportional band limits of control output.  
Determined by value of  $K_p$ .

At higher  $K_p$ ,  $e$  produces more correction but reduces the range between controller limits



# Finding $K_p$ and the Proportional Band

$$K_p = \frac{\Delta \text{Output}}{\text{SP} - \text{Measurement}}$$

$$K_p = \frac{\Delta \text{Output}}{\Delta e}$$

Proportional gain - change in output / change in error

Increasing proportional gain decreases proportional band (PB). In terms of percent.

$$\%PB = \frac{1}{K_p} \cdot 100\%$$

$\%PB$  inversely proportional to  $K_p$

Example:  $\Delta \text{output} \pm 15 \text{Vdc}$  OPAMP Limits Fixed by saturation

$$\Delta e = \pm 3 \text{Vdc}$$

Find  $K_p$  and  $\%PB$

$$K_p = \frac{e_{\max} - e_{\min}}{o_{\max} - o_{\min}} = \frac{15 - (-15)}{3 - (-3)} = \frac{30}{6} = 5 = \frac{\Delta \text{Output}}{\Delta e}$$

$$\%PB = \frac{1}{K_p} 100\% = \frac{1}{5} (100\%) = 20\%$$

Based on  $C_o$ , 20% change in  $\Delta e$  Gives Full  $\Delta$  output

$$\Delta \text{output} (0.20) = \Delta e \Rightarrow 30 (0.2) = 6$$

Set  $K_p = 2$  find  $\%PB$  and max  $\Delta$  output  $\%PB = \frac{1}{2} (100\%) = 50\%$

$$K_p = \frac{\Delta \text{output}}{\Delta e} \Rightarrow \Delta e K_p = \Delta \text{output} \Rightarrow 6 (2) = \pm 12 \text{V output range}$$

Greater error can be handled before saturation

# Proportional Gain and Proportional Band

## Example. Continued

Decreasing %PB (increasing  $K_p$ ) causes a smaller error signal to cause the controller to reach its limits

ORIGINAL %PB = 20%  $\pm 6V_{dc}$  causes  $C_0$  to vary  $\pm 15V_{dc}$

Decrease %PB to 10%

A VARIATION OF  $\frac{1}{2}$  OF ORIGINAL  $\Delta e$  Causes  $C_0$  to REACH LIMITS

$$\%PB = \frac{1}{K_p} 100\% \Rightarrow \frac{10\%}{100\%} = \frac{1}{K_p} \Rightarrow K_p = 10$$

Find  $\Delta e$  input that produces max  $\Delta$  output

$$\Delta_{\text{output}} = 30 \text{ MAX}$$

$$\frac{\Delta C_0}{\Delta e} = K_p = \frac{\Delta_{\text{output}}}{\Delta e} = \Delta e K_p = \Delta_{\text{output}}$$

Solve for  $\Delta e$

$$\Delta e = \frac{\Delta_{\text{output}}}{K_p}$$

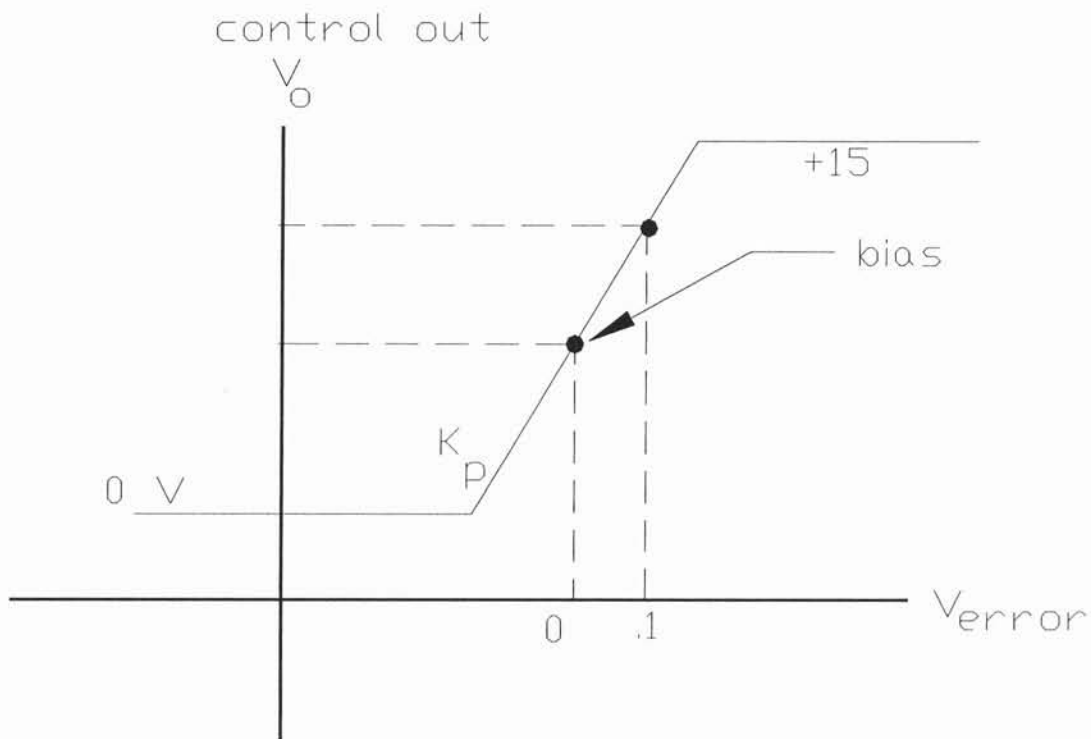
$$\Delta e = \frac{30V}{10} = \pm 3V (\pm 1.5)$$

Proportion changes



# Characteristics of Proportional Control

$$C_o = K_p e + C_b \quad \text{Control formula}$$



If  $e = 0$  then  $C_o = C_b$

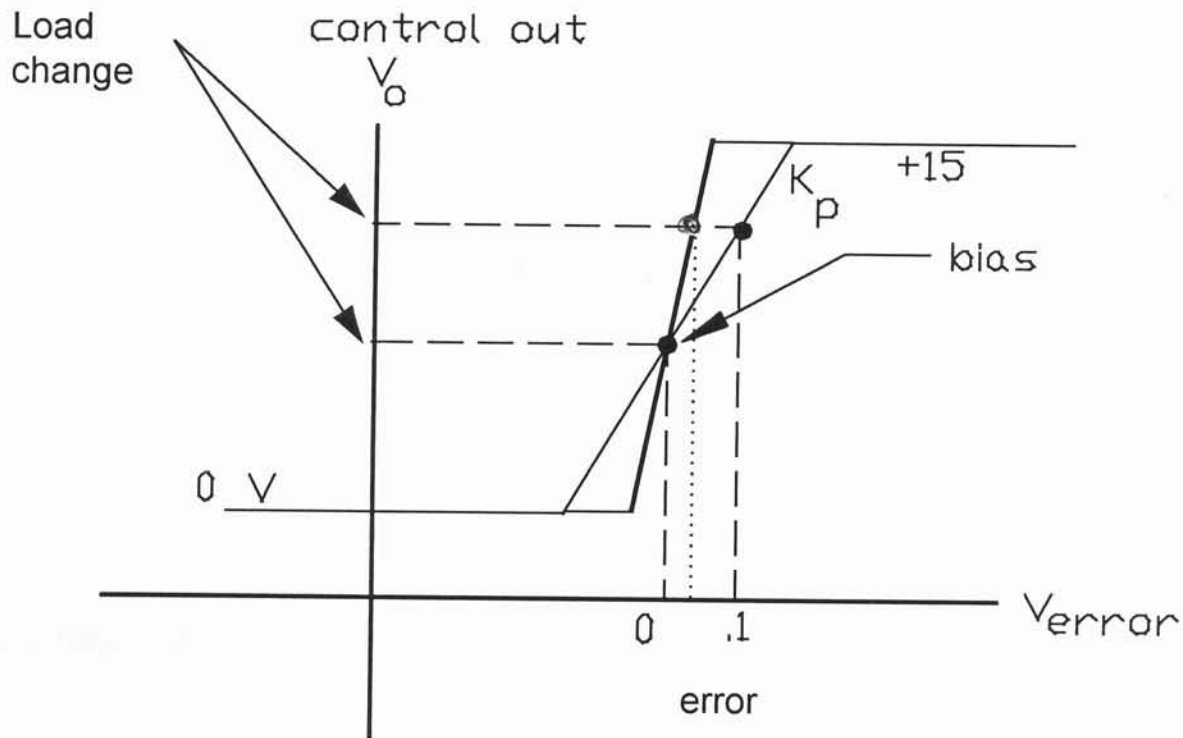
If  $e$  not = 0, then for every unit of error  $K_p(e)$  units of correction are added/subtracted to control output.

There is a band of steady-state error about 0 that has a magnitude of  $PB\%$  where the output is not saturated.

# Offset and residual error

residual error - also known as steady-state error.

Proportional control always produces a steady-state or residual error when a change in the process occurs



If the load change is permanent, then the error is permanent. Increasing  $K_p$  decreases steady-state error but cannot eliminate it.

If load change is transient, then error returns to bias value, 0.

# Effects of Increased Gain on error and stability.

High gain reduces steady-state error but increases chances of instability

## Example

$$\text{FOR } K_p = 2 \quad V_o = 0 - 15 \text{ Vdc} = C_o$$

$$V_e = 0 \pm 1 \text{ Vdc} = e$$

$$V_{\text{bias}} = 7.5 \text{ Vdc} = C_b$$

$$V_o = K_p V_e + V_{\text{bias}} \Rightarrow C_o = K_p e + C_b \Rightarrow V_o = 2V_e + 7.5 \text{ V}$$

$V_e = 0$  change

$$V_o = 2(0) + 7.5 = 7.5 \text{ with no steady-state error}$$

Process changes  $V_e = -0.25$

$$V_o = 2(-0.25) + 7.5 = 7.0 \text{ V}$$

Find Residual error Remember  $K_p = \frac{C_o}{e} \Rightarrow e = \frac{C_o}{K_p}$

error is inversely Proportional to  $K_p$

$$\frac{V_o}{K_p} = V_e = \frac{7.0}{2} = 3.5 \text{ V}$$

For Control error to reach 0  $K_p \rightarrow \infty$  Set  $K_p = 5$

Calculate

$$\frac{V_o}{K_p} = \frac{7.0}{5} = 1.4 \text{ V}$$